

# 六维系统环形桁架天线的非线性动力学分析\*

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**摘要:** 随着科技的发展,大尺度、低重量、易收拢、高精度等特点是未来天线的主要发展方向.环形桁架天线在发射时整体处于收拢状态,升空后按指令有顺序展开,节省了航天器的空间.此外,环形桁架天线可根据需求设计展开口径的大小,所以,环形桁架天线是目前较为理想的天线结构形式.由于自身结构特点以及复杂的空间环境因素,天线在运行时易产生大幅度的非线性振动,严重影响卫星的稳定运行.因此,将环形桁架天线简化成等效圆柱壳模型,并建立其动力学方程.采用理论分析和数值模拟研究了六维系统环形桁架天线的非线性动力学特性.利用规范型理论化简系统方程分析未扰系统和扰动系统的非线性动力学行为,利用能量相位法验证环形桁架天线系统具有 Shilnikov 型多脉冲混沌运动,利用数值模拟验证理论分析,并通过数值模拟研究了热激励对环形桁架天线系统非线性振动的影响.

**关键词:** 环形桁架天线; 规范型; 能量相位法; 混沌运动

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## 引言

人造卫星技术是航天科技领域中的关键组成部分.伴随航天科技的飞速发展以及为满足国防和民用等各领域的需求,近些年来,人造卫星技术也在不断地发展.星载天线负责卫星信号的接收与传递,是卫星的核心组成部分.大尺度、高精度和低成本等特点是近些年也是未来卫星天线的主要发展方向<sup>[1-2]</sup>.卫星进入轨道后天线在随卫星运行过程中由于尺寸大以及材料柔韧性等自身结构特点,外加复杂的太空环境,如太阳辐射、热激励和微重力等因素,将导致天线产生大幅的非线性振动,严重影响卫星姿态的稳定以及信号精度的控制<sup>[3-4]</sup>.因此,卫星天线非线性振动的研究受到国内外许多学者的关注.

Ghosh 和 Kumar<sup>[5]</sup>利用有限元方法分析了可移动支撑类型天线在激励作用下产生的非线性响应.Makarov 等<sup>[6]</sup>研究了失稳情形下环形柔性可展天线的非线性动力学行为.赵孟良和关富玲<sup>[7]</sup>建立了含摩擦的周边桁架式环形天线的动力学模型,分析了桁架单元之间的铰链摩擦以及重力作用等因素对桁架展开的影响.他们设计了四面体构架式的桁架可展开天线、六棱柱单元构架式可展开天线原理样机,并深入研究了所研制的天线结构设计、展开控制、运动特性、动力学特性<sup>[8-9]</sup>等.胡海岩等<sup>[10-11]</sup>综述了大型可展天线的研究现状,提出了大型可展天线在结

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构设计、动力学建模与控制 and 仿真模拟实验中需要注意的问题,建立了大型可展天线展开后在空中运行的动力学模型并分析了其非线性动力学行为.文献[12]基于 Lagrange(拉格朗日)方程建立了可展桁架天线的柔性多体动力学模型,讨论了索网张力对天线桁架的影响,并进行了动态展开分析.Gao 等<sup>[13]</sup>研究了安装两个刚性支承臂的柔性空间天线在 3:1 和 2:1 的内共振情形下的近似解及分叉.Pellicano<sup>[14]</sup>利用 Sanders-Koiter 理论将系统简化成圆柱壳并建立了其非线性动力学方程,分析了系统在压缩静力和周期性轴向载荷综合作用下的稳定性.文献[15]将环形桁架卫星天线等效成圆柱壳结构,建立等效圆柱壳结构的呼吸振动形式下的非线性动力学控制方程,并分析了热激励对系统非线性动力学行为的影响.

为研究环形桁架天线在运行中的非线性动力学行为,将环形桁架天线结构简化成力学模型并建立其动力学方程,所建立的动力学方程都可以用高维非线性系统来描述.因此,研究高维非线性动力学方程,不仅对分析卫星天线的非线性动力学行为有重要意义,而且对于研究由工程实际问题所建立的高维非线性动力学方程也具有指导意义.对于高维非线性系统混沌运动的研究从最早的符号动力学、Smale 马蹄运动等到现在的能量相位法<sup>[16-17]</sup>、广义 Melnikov 方法<sup>[18-19]</sup>以及数值方法等,研究方法和理论都在不断改进,从而更广泛地适用于研究实际工程问题.目前,为了研究工程实际问题中高维非线性系统的多脉冲混沌运动,研究者将能量相位法不断推广和改进<sup>[20-22]</sup>.对于实际工程问题的动力学特性研究还有一些其他的方法,如赵岩等<sup>[23]</sup>利用虚拟激励法研究了受随机载荷作用下陀螺阻尼系统随机动力响应问题.黎岫珉等<sup>[24]</sup>在随机激励环境中研究刚度非线性和阻尼非线性对隔振系统在随机激励下力传递率及其概率分布的影响.吴子英等<sup>[25]</sup>建立了附加线性振子的双稳态电磁式振动能量捕获器的动力学方程,并研究有色噪声激励作用下双稳态能量捕获系统的非线性动力学行为.

卫星天线在轨运行时受到太阳辐射的作用,天线表面在受照区和阴影区的交替过程中温度发生剧烈的冷热交变,引发天线结构热变形、热应力等.因此,环形可展天线的外激励主要为热激励.本文考虑以横向振动为主要振动,对环形桁架天线的动力学方程<sup>[15]</sup>进行三阶离散,得到系统以呼吸振动为主要振动的三自由度非线性方程.对所得到的三自由度非线性方程即六维非线性方程,选取 1:4:6 内共振关系,将推广后适用于分析六维系统的能量相位法<sup>[22]</sup>进一步地化简并研究环形桁架天线的多脉冲混沌动力学.

## 1 环形桁架天线的非线性动力学方程和规范型化简

### 1.1 环形桁架天线的非线性动力学方程

在文献[15]中环形桁架天线被简化成如图 1 所示的等效圆柱壳模型, $R$  为圆柱壳的面半径, $L$  为轴向长度, $h$  为沿径向到壳的厚度.圆柱壳处于变化温度场中, $\Delta T = T_1 \cos(\Omega t) - T_{\text{ref}}$ , $T_{\text{ref}}$  为初始温度, $T_1 \cos(\Omega t)$  为周期性的热激励扰动项.

根据 Reddy 一阶剪切变形壳理论、Von Karman 非线性应力-应变关系以及 Hamilton 原理,建立了考虑剪切的等效圆柱壳的非线性动力学方程:

$$\begin{aligned}
 & A_{11} \frac{\partial^2 u_0}{\partial x^2} + A_{66} \frac{1}{R^2} \frac{\partial^2 u_0}{\partial \theta^2} + (A_{12} + A_{66}) \frac{1}{R} \frac{\partial^2 v_0}{\partial x \partial \theta} + B_{66} \frac{1}{R^2} \frac{\partial^2 \varphi_x}{\partial \theta^2} + (B_{12} + B_{66}) \frac{1}{R} \frac{\partial^2 \varphi_\theta}{\partial x \partial \theta} + \\
 & A_{11} \frac{\partial w_0}{\partial x} \frac{\partial^2 w_0}{\partial x^2} + A_{66} \frac{1}{R^2} \frac{\partial w_0}{\partial x} \frac{\partial^2 w_0}{\partial \theta^2} + (A_{12} + A_{66}) \frac{1}{R^2} \frac{\partial w_0}{\partial \theta} \frac{\partial^2 w_0}{\partial x \partial \theta} + \frac{A_{12}}{R} \frac{\partial w_0}{\partial x} + \\
 & B_{11} \frac{\partial^2 \varphi_x}{\partial x^2} = G_0 \ddot{u}_0 + G_1 \ddot{\varphi}_x, \tag{1a}
 \end{aligned}$$

$$\begin{aligned}
& A_{66} \frac{\partial^2 v_0}{\partial x^2} + A_{22} \frac{1}{R^2} \frac{\partial^2 v_0}{\partial \theta^2} + (A_{12} + A_{66}) \frac{1}{R} \frac{\partial^2 u_0}{\partial x \partial \theta} + B_{22} \frac{1}{R^2} \frac{\partial^2 \varphi_\theta}{\partial \theta^2} + (B_{12} + B_{66}) \frac{1}{R} \frac{\partial^2 \varphi_x}{\partial x \partial \theta} + \\
& B_{66} \frac{\partial^2 \varphi_\theta}{\partial x^2} + A_{66} \frac{1}{R} \frac{\partial w_0}{\partial \theta} \frac{\partial^2 w_0}{\partial x^2} + A_{22} \frac{1}{R^3} \frac{\partial w_0}{\partial \theta} \frac{\partial^2 w_0}{\partial \theta^2} + (A_{12} + A_{66}) \frac{1}{R} \frac{\partial w_0}{\partial x} \frac{\partial^2 w_0}{\partial x \partial \theta} + \\
& \left( \frac{A_{22}}{R} + \frac{KA_{44}}{R} \right) \frac{1}{R} \frac{\partial w_0}{\partial \theta} + \frac{K}{R} A_{44} \left( \varphi_\theta - \frac{v_0}{R} \right) = G_0 \ddot{v}_0 + G_1 \ddot{\varphi}_\theta, \quad (1b)
\end{aligned}$$

$$\begin{aligned}
& A_{11} \frac{\partial u_0}{\partial x} \frac{\partial^2 w_0}{\partial x^2} + A_{12} \frac{1}{R^2} \frac{\partial u_0}{\partial x} \frac{\partial^2 w_0}{\partial \theta^2} + 2A_{66} \frac{1}{R^2} \frac{\partial u_0}{\partial \theta} \frac{\partial^2 w_0}{\partial x \partial \theta} + (A_{12} + A_{66}) \frac{1}{R^2} \frac{\partial^2 u_0}{\partial x \partial \theta} \frac{\partial w_0}{\partial \theta} + \\
& A_{66} \frac{1}{R^2} \frac{\partial^2 u_0}{\partial \theta^2} \frac{\partial w_0}{\partial x} - \frac{A_{12}}{R} \frac{\partial u_0}{\partial x} + (A_{12} + A_{66}) \frac{1}{R} \frac{\partial^2 v_0}{\partial x \partial \theta} \frac{\partial w_0}{\partial x} - \frac{(A_{22} + KA_{44})}{R} \frac{1}{R} \frac{\partial v_0}{\partial \theta} + \\
& 2A_{66} \frac{1}{R} \frac{\partial v_0}{\partial x} \frac{\partial^2 w_0}{\partial x \partial \theta} + A_{22} \frac{1}{R^3} \frac{\partial v_0}{\partial \theta} \frac{\partial^2 w_0}{\partial \theta^2} + A_{12} \frac{1}{R} \frac{\partial v_0}{\partial \theta} \frac{\partial^2 w_0}{\partial x^2} + A_{22} \frac{1}{R^3} \frac{\partial^2 v_0}{\partial \theta^2} \frac{\partial w_0}{\partial \theta} + \\
& \left( KA_{55} - \frac{B_{21}}{R} \right) \frac{\partial \varphi_x}{\partial x} + B_{12} \frac{1}{R^2} \frac{\partial \varphi_x}{\partial x} \frac{\partial^2 w_0}{\partial \theta^2} + 2B_{66} \frac{1}{R^2} \frac{\partial \varphi_x}{\partial \theta} \frac{\partial^2 w_0}{\partial x \partial \theta} + B_{11} \frac{\partial^2 \varphi_x}{\partial x^2} \frac{\partial w_0}{\partial x} + \\
& (B_{12} + B_{66}) \frac{1}{R^2} \frac{\partial^2 \varphi_x}{\partial x \partial \theta} \frac{\partial w_0}{\partial \theta} + B_{11} \frac{\partial \varphi_x}{\partial x} \frac{\partial^2 w_0}{\partial x^2} + B_{66} \frac{1}{R^2} \frac{\partial^2 \varphi_x}{\partial \theta^2} \frac{\partial w_0}{\partial x} - \frac{A_{22} w}{R^2} + \\
& A_{11} \frac{\partial^2 u_0}{\partial x^2} \frac{\partial w_0}{\partial x} + \left( KA_{44} - \frac{B_{22}}{R} \right) \frac{1}{R} \frac{\partial \varphi_\theta}{\partial \theta} + B_{66} \frac{1}{R} \frac{\partial^2 \varphi_\theta}{\partial x^2} \frac{\partial w_0}{\partial \theta} + B_{22} \frac{1}{R^3} \frac{\partial \varphi_\theta}{\partial \theta} \frac{\partial^2 w_0}{\partial \theta^2} + \\
& A_{66} \frac{1}{R} \frac{\partial^2 v_0}{\partial x^2} \frac{\partial w_0}{\partial \theta} + B_{12} \frac{1}{R} \frac{\partial \varphi_\theta}{\partial \theta} \frac{\partial^2 w_0}{\partial x^2} + \frac{3}{2} A_{11} \left( \frac{\partial w_0}{\partial x} \right)^2 \frac{\partial^2 w_0}{\partial x^2} + 2B_{66} \frac{1}{R} \frac{\partial \varphi_\theta}{\partial x} \frac{\partial^2 w_0}{\partial x \partial \theta} + \\
& B_{22} \frac{1}{R^3} \frac{\partial^2 \varphi_\theta}{\partial \theta^2} \frac{\partial w_0}{\partial \theta} + 2(A_{12} + 2A_{66}) \frac{1}{R} \frac{\partial w_0}{\partial x} \frac{\partial w_0}{\partial \theta} \frac{\partial^2 w_0}{\partial x^2} + (B_{12} + B_{66}) \frac{1}{R} \frac{\partial^2 \varphi_\theta}{\partial x \partial \theta} \frac{\partial w_0}{\partial x} + \\
& \frac{A_{12} \left( \frac{\partial w_0}{\partial x} \right)^2}{2R} + \frac{3}{2} A_{22} \frac{1}{R^4} \left( \frac{\partial w_0}{\partial \theta} \right)^2 \frac{\partial^2 w_0}{\partial \theta^2} + \left( \frac{A_{12} w}{R} + KA_{55} \right) \frac{\partial^2 w_0^2}{\partial x^2} + \frac{2N_{x\theta}^T}{R} \frac{\partial^2 w_0}{\partial x \partial \theta} + \\
& \left( \frac{A_{22} w}{R} + KA_{44} \right) \frac{1}{R^2} \frac{\partial^2 w_0}{\partial \theta^2} + \left( \frac{1}{2} A_{12} + A_{66} \right) \frac{1}{R^2} \left( \frac{\partial w_0}{\partial x} \right)^2 \frac{\partial^2 w_0}{\partial \theta^2} + \frac{A_{22}}{2R} \frac{1}{R^2} \left( \frac{\partial w_0}{\partial \theta} \right) + \\
& \left( \frac{1}{2} A_{12} + A_{66} \right) \frac{1}{R^2} \left( \frac{\partial w_0}{\partial \theta} \right)^2 \frac{\partial^2 w_0}{\partial x^2} + N_{xx}^T \frac{\partial^2 w_0}{\partial x^2} + \frac{1}{R^2} N_{yy}^T \frac{\partial^2 w_0}{\partial \theta^2} - \mu \frac{\partial w_0}{\partial t} - \\
& \frac{N_{yy}^T}{R} = G_0 \ddot{w}_0, \quad (1c)
\end{aligned}$$

$$\begin{aligned}
& B_{11} \frac{\partial^2 u_0}{\partial x^2} + B_{66} \frac{1}{R^2} \frac{\partial^2 u_0}{\partial \theta^2} + (B_{12} + B_{66}) \frac{1}{R} \frac{\partial^2 v_0}{\partial x \partial \theta} + C_{66} \frac{1}{R^2} \frac{\partial^2 \varphi_x}{\partial \theta^2} + (C_{12} + C_{66}) \frac{1}{R} \frac{\partial^2 \varphi_\theta}{\partial x \partial \theta} + \\
& C_{11} \frac{\partial^2 \varphi_x}{\partial x^2} + B_{11} \frac{\partial w_0}{\partial x} \frac{\partial^2 w_0}{\partial x^2} + (B_{12} + B_{66}) \frac{1}{R^2} \frac{\partial w_0}{\partial \theta} \frac{\partial^2 w_0}{\partial x \partial \theta} + B_{66} \frac{1}{R^2} \frac{\partial w_0}{\partial x} \frac{\partial^2 w_0}{\partial \theta^2} - \\
& KA_{55} \varphi_x + \left( \frac{B_{12}}{R} - KA_{55} \right) \frac{\partial w_0}{\partial x} = G_1 \ddot{u}_0 + G_2 \ddot{\varphi}_x, \quad (1d)
\end{aligned}$$

$$B_{66} \frac{\partial^2 v_0}{\partial x^2} + B_{22} \frac{1}{R^2} \frac{\partial^2 v_0}{\partial \theta^2} + (B_{12} + B_{66}) \frac{1}{R} \frac{\partial^2 u_0}{\partial x \partial \theta} + C_{66} \frac{\partial^2 \varphi_\theta}{\partial x^2} + C_{22} \frac{1}{R^2} \frac{\partial^2 \varphi_\theta}{\partial \theta^2} + \frac{B_{22}}{R^2} \frac{\partial w_0}{\partial \theta} +$$

$$\begin{aligned} & (C_{12} + C_{66}) \frac{1}{R} \frac{\partial^2 \varphi_x}{\partial x \partial \theta} + B_{66} \frac{1}{R} \frac{\partial w_0}{\partial \theta} \frac{\partial^2 w_0}{\partial x^2} + (B_{12} + B_{66}) \frac{1}{R} \frac{\partial w_0}{\partial x} \frac{\partial^2 w_0}{\partial x \partial \theta} - \\ & KA_{44} \frac{1}{R} \frac{\partial w}{\partial \theta} + B_{22} \frac{1}{R^3} \frac{\partial w_0}{\partial \theta} \frac{\partial^2 w_0}{\partial \theta^2} - KA_{44} \left( \varphi_\theta - \frac{v_0}{R} \right) = G_1 \ddot{v}_0 + G_0 \ddot{\varphi}_\theta, \end{aligned} \quad (1e)$$

其中  $A_{ij}, B_{ij}, C_{ij} (i, j = 1, 2, \dots, 6)$  分别代表等效圆柱壳的薄膜刚度、耦合刚度和弯曲刚度,且被定义为

$$A_{ij}, B_{ij}, C_{ij} = \sum_{k=1}^{10} \int_{z_k}^{z_{k+1}} Q_{ij}(1, z, z^2) dz, \quad i, j = 1, 2, 6, \quad (2a)$$

$$A_{ij} = \sum_{k=1}^{10} \int_{z_k}^{z_{k+1}} Q_{i,j}(1, z, z^2) dz, \quad i, j = 4, 5; \quad (2b)$$

$$G_i = \int_{-h/2}^{h/2} \rho(1, z, z^2) dz, \quad i = 0, 1, 2.$$

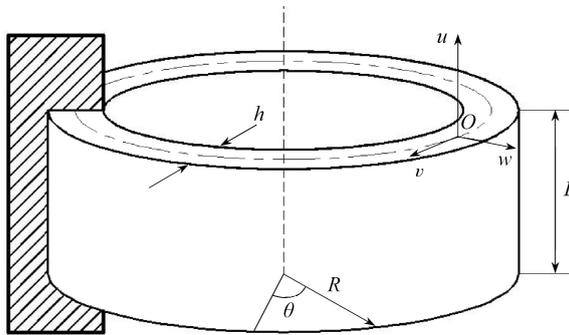


图1 等效圆柱壳模型

Fig. 1 The model for the equivalent circular cylindrical shell

考虑如下自由-自由以及固定-固定边界条件:

固定-固定端

$$v_0 = w_0 = \varphi_\theta = 0, \quad \theta = 0, 2\pi; \quad (3a)$$

自由-自由端

$$N_{xx} = N_{xy} = M_{xx} = M_{xy} = Q_x = 0, \quad x = 0, L. \quad (3b)$$

为便于接下来的定性分析和定量分析不受量纲的影响,对方程进行无量纲化.引入变量和参数的无量纲形式:

$$\begin{aligned} \bar{u}_0 &= \frac{u_0}{L}, \quad \bar{v}_0 = \frac{v_0}{R}, \quad \bar{w}_0 = \frac{w_0}{h}, \quad \bar{\varphi}_x = \varphi_x, \quad \bar{\varphi}_y = \varphi_y, \quad \bar{x} = \frac{x}{L}, \\ \bar{\Omega} &= \frac{1}{\pi^2} \left( \frac{LR\rho}{E} \right)^{1/2} \Omega, \quad \bar{t} = \pi^2 \left( \frac{E}{LR\rho} \right)^{1/2} t, \quad \bar{\mu} = \frac{(LR)^2}{\pi^2 h^4} \left( \frac{1}{\rho E} \right)^{1/2} \mu, \quad \bar{A}_{ij} = \frac{(LR)^{1/2}}{Eh^2} A_{ij}, \\ \bar{B}_{ij} &= \frac{(LR)^{1/2}}{Eh^3} B_{ij}, \quad \bar{C}_{ij} = \frac{(LR)^{1/2}}{Eh^4} C_{ij}, \quad \bar{G}_i = \frac{1}{(LR)^{(i+1)/2} \rho} G_i. \end{aligned}$$

考虑以横向振动为主要振动,对方程(1)的  $w$  方向进行三阶离散,其余方向一阶离散,得到系统由横向位移表示的呼吸振动的三自由度非线性方程如下:

$$\begin{aligned} \ddot{w}_1 + \mu_1 \dot{w}_1 + \omega_1^2 w_1 + f_1 \cos(\Omega t) w_1 + \alpha_{11} w_1^2 + \alpha_{12} w_2^2 + \alpha_{13} w_3^2 + \alpha_{14} w_1^3 + \alpha_{15} w_2^3 + \\ \alpha_{16} w_3^3 + \alpha_{17} w_1 w_2 + \alpha_{18} w_2 w_3 + \alpha_{19} w_3 w_1 + \alpha_{20} w_1^2 w_2 + \alpha_{21} w_2^2 w_1 + \alpha_{22} w_2^2 w_3 + \\ \alpha_{23} w_3^2 w_2 + \alpha_{24} w_1^2 w_3 + \alpha_{25} w_3^2 w_1 + \alpha_{26} w_1 w_2 w_3 = F_1 \cos(\Omega_1 t), \end{aligned} \quad (4a)$$

$$\ddot{w}_2 + \mu_2 \dot{w}_2 + \omega_2^2 w_2 + f_2 \cos(\Omega t) w_2 + \beta_{11} w_1^2 + \beta_{12} w_2^2 + \beta_{13} w_3^2 + \beta_{14} w_1^3 + \beta_{15} w_2^3 + \beta_{16} w_3^3 + \beta_{17} w_1 w_2 + \beta_{18} w_2 w_3 + \beta_{19} w_3 w_1 + \beta_{20} w_1^2 w_2 + \beta_{21} w_2^2 w_1 + \beta_{22} w_2^2 w_3 + \beta_{23} w_3^2 w_2 + \beta_{24} w_1^2 w_3 + \beta_{25} w_3^2 w_1 + \beta_{26} w_1 w_2 w_3 = F_2 \cos(\Omega_2 t), \quad (4b)$$

$$\ddot{w}_3 + \mu_3 \dot{w}_3 + \omega_3^2 w_3 + f_3 \cos(\Omega t) w_3 + \gamma_{11} w_1^2 + \gamma_{12} w_2^2 + \gamma_{13} w_3^2 + \gamma_{14} w_1^3 + \gamma_{15} w_2^3 + \gamma_{16} w_3^3 + \gamma_{17} w_1 w_2 + \gamma_{18} w_2 w_3 + \gamma_{19} w_3 w_1 + \gamma_{20} w_1^2 w_2 + \gamma_{21} w_2^2 w_1 + \gamma_{22} w_2^2 w_3 + \gamma_{23} w_3^2 w_2 + \gamma_{24} w_1^2 w_3 + \gamma_{25} w_3^2 w_1 + \gamma_{26} w_1 w_2 w_3 = F_3 \cos(\Omega_3 t), \quad (4c)$$

其中  $w_1(t)$ ,  $w_2(t)$  和  $w_3(t)$  分别是第一阶、第二阶和第三阶振动的模式,  $\mu_1, \mu_2$  和  $\mu_3$  为结构阻尼系数,  $f_1, f_2, f_3$  为环形桁架天线的参数激励,  $F_1, F_2$  和  $F_3$  为环形桁架天线的外激励, 方程的其他系数见文献[15].

利用多尺度法分析等效圆柱壳的非线性呼吸振动方程, 考虑如下 1:4:6 内共振情形:

$$\omega_1^2 = \frac{1}{4} \Omega_1^2 + \varepsilon \sigma_1, \quad \omega_2^2 = 4\Omega_2^2 + \varepsilon \sigma_2, \quad \omega_3^2 = 6\Omega_3^2 + \varepsilon \sigma_3, \quad (5)$$

其中  $\sigma_1, \sigma_2$  和  $\sigma_3$  为三个调谐参数, 为简化分析, 令

$$\Omega = \Omega_1 = \frac{1}{2} \Omega_2 = \frac{1}{3} \Omega_3 = 1.$$

经过计算得到直角坐标系下的平均方程:

$$\begin{aligned} \dot{x}_1 = & -\frac{1}{2} \mu_1 x_1 - (\sigma_1 + f_1) x_2 - \frac{9}{4} \alpha_{14} x_2 (x_1^2 + x_2^2) - \\ & \frac{1}{2} \alpha_{21} x_2 (x_3^2 + x_4^2) - \frac{1}{2} \alpha_{25} x_2 (x_5^2 + x_6^2), \end{aligned} \quad (6a)$$

$$\begin{aligned} \dot{x}_2 = & (\sigma_1 - f_1) x_1 - \frac{1}{2} \mu_1 x_2 + \frac{9}{4} \alpha_{14} x_1 (x_1^2 + x_2^2) + \\ & \frac{1}{2} \alpha_{21} x_1 (x_3^2 + x_4^2) + \frac{1}{2} \alpha_{25} x_1 (x_5^2 + x_6^2), \end{aligned} \quad (6b)$$

$$\begin{aligned} \dot{x}_3 = & -\frac{1}{2} \mu_2 x_3 - \frac{1}{4} \sigma_2 x_4 - \frac{9}{16} \beta_{15} x_4 (x_3^2 + x_4^2) - \frac{1}{8} \beta_{20} x_4 (x_1^2 + x_2^2) - \\ & \frac{1}{8} \beta_{23} x_4 (x_5^2 + x_6^2) - \frac{1}{8} \beta_{24} x_1 x_2 x_5 - \frac{1}{16} \beta_{24} x_6 (x_1^2 - x_2^2), \end{aligned} \quad (6c)$$

$$\begin{aligned} \dot{x}_4 = & \frac{1}{4} \sigma_2 x_3 - \frac{1}{2} \mu_2 x_4 + \frac{9}{16} \beta_{15} x_3 (x_3^2 + x_4^2) + \frac{1}{8} \beta_{20} x_3 (x_1^2 + x_2^2) + \\ & \frac{1}{8} \beta_{23} x_3 (x_5^2 + x_6^2) - \frac{1}{8} \beta_{24} x_1 x_2 x_6 + \frac{1}{16} \beta_{24} x_5 (x_1^2 - x_2^2) - \frac{1}{4} F_2, \end{aligned} \quad (6d)$$

$$\begin{aligned} \dot{x}_5 = & -\frac{1}{2} \mu_3 x_5 - \frac{1}{6} \sigma_3 x_6 - \frac{3}{8} \gamma_{16} x_6 (x_5^2 + x_6^2) - \frac{1}{12} \gamma_{20} x_1 x_2 x_3 - \\ & \frac{1}{24} \gamma_{20} x_4 (x_1^2 - x_2^2) - \frac{1}{12} \gamma_{22} x_6 (x_3^2 + x_4^2) - \frac{1}{12} \gamma_{24} x_6 (x_1^2 + x_2^2), \end{aligned} \quad (6e)$$

$$\begin{aligned} \dot{x}_6 = & \frac{1}{6} \sigma_3 x_5 - \frac{1}{2} \mu_3 x_6 + \frac{3}{8} \gamma_{16} x_5 (x_5^2 + x_6^2) - \frac{1}{12} \gamma_{20} x_1 x_2 x_4 + \\ & \frac{1}{24} \gamma_{20} x_3 (x_1^2 + x_2^2) + \frac{1}{12} \gamma_{22} x_5 (x_3^2 + x_4^2) + \frac{1}{12} \gamma_{24} x_5 (x_1^2 + x_2^2) - \frac{1}{6} F_3. \end{aligned} \quad (6f)$$

## 1.2 环形桁架天线动力学方程的规范型化简

由于环形桁架天线的平均方程(6)中所含的非线性项复杂,不能对其非线性方程进行直接分析.因此,利用规范型理论对复杂的非线性方程进行化简,而且化简后的方程与原方程是拓扑等价的.

系统(6)在初始平衡点  $(x_1, x_2, x_3, x_4, x_5, x_6) = (0, 0, 0, 0, 0, 0)$  线性部分的 Jacobi 矩阵为

$$J = \begin{bmatrix} -\frac{1}{2}\mu_1 & -(\sigma_1 + f_1) & 0 & 0 & 0 & 0 \\ \sigma_1 - f_1 & -\frac{1}{2}\mu_1 & 0 & 0 & 0 & 0 \\ 0 & 0 & -\frac{1}{2}\mu_2 & -\frac{1}{2}\sigma_2 & 0 & 0 \\ 0 & 0 & \frac{1}{2}\sigma_2 & -\frac{1}{2}\mu_2 & 0 & 0 \\ 0 & 0 & 0 & 0 & -\frac{1}{2}\mu_3 & -\frac{1}{2}\sigma_3 \\ 0 & 0 & 0 & 0 & \frac{1}{2}\sigma_3 & -\frac{1}{2}\mu_3 \end{bmatrix}. \quad (7)$$

Jacobi 矩阵对应的特征多项式为

$$f(\lambda) = \left( \lambda^2 + \lambda\mu_1 + \frac{1}{4}\mu_1^2 + \sigma_1^2 - f_1^2 \right) \times \left( \lambda^2 + \lambda\mu_2 + \frac{1}{4}\mu_2^2 + \frac{1}{16}\sigma_2^2 \right) \left( \lambda^2 + \lambda\mu_3 + \frac{1}{4}\mu_3^2 + \frac{1}{16}\sigma_3^2 \right). \quad (8)$$

由方程(8)可知,当  $\mu_1 = \mu_2 = \mu_3 = 0, \sigma_1^2 - f_1^2 = 0$  时,对应的环形桁架天线的平均方程(6)有一对零特征根和两对纯虚特征根:

$$\lambda_{1,2} = 0, \lambda_{3,4} = \pm \frac{1}{4}\sigma_2 i, \lambda_{5,6} = \pm \frac{1}{4}\sigma_3 i. \quad (9)$$

令  $f_1 = -1, \sigma_1 = f_1 + \bar{\sigma}_1$ , 将  $\mu_1, \mu_2, \mu_3, \bar{\sigma}_1, F_2, F_3$  作为摄动参数,利用 MAPLE 程序得到系统(6)含参数的三阶规范型:

$$\dot{y}_1 = -\frac{1}{2}\mu_1 y_1 + (1 - \bar{\sigma}_1) y_2, \quad (10a)$$

$$\begin{aligned} \dot{y}_2 = & \bar{\sigma}_1 y_1 - \frac{1}{2}\mu_1 y_2 + \frac{9}{4}\alpha_{14} y_1^3 + \frac{1}{2}\alpha_{21} y_1 y_3^2 + \\ & \frac{1}{2}\alpha_{21} y_1 y_4^2 + \frac{1}{2}\alpha_{25} y_1 y_5^2 + \frac{1}{2}\alpha_{25} y_1 y_6^2, \end{aligned} \quad (10b)$$

$$\begin{aligned} \dot{y}_3 = & -\frac{1}{2}\mu_2 y_3 - \frac{1}{4}\sigma_2 y_4 + \frac{9}{128}\beta_{15} y_3^3 - \frac{63}{128}\beta_{15} y_3^2 y_4 - \frac{1}{8}\beta_{20} y_1^2 y_4 - \\ & \frac{63}{128}\beta_{15} y_3^2 y_4 + \frac{9}{128}\beta_{15} y_3 y_4^2 + \frac{1}{16}\beta_{23} y_3 y_5^2 - \frac{1}{16}\beta_{23} y_4 y_5^2 + \\ & \frac{1}{16}\beta_{23} y_3 y_6^2 - \frac{1}{16}\beta_{23} y_4 y_6^2, \end{aligned} \quad (10c)$$

$$\dot{y}_4 = \frac{1}{4}\sigma_2 y_3 - \frac{1}{2}\mu_2 y_4 + \frac{63}{128}\beta_{15} y_3^3 + \frac{9}{128}\beta_{15} y_3^2 y_4 + \frac{1}{8}\beta_{20} y_1^2 y_3 +$$

$$\begin{aligned} & \frac{9}{128}\beta_{15}y_3^2y_4 + \frac{63}{128}\beta_{15}y_3y_4^2 + \frac{1}{16}\beta_{23}y_3y_5^2 + \frac{1}{16}\beta_{23}y_4y_5^2 + \\ & \frac{1}{16}\beta_{23}y_3y_6^2 + \frac{1}{16}\beta_{23}y_4y_6^2 - \frac{1}{4}F_2, \end{aligned} \quad (10d)$$

$$\begin{aligned} \dot{y}_5 = & -\frac{1}{2}\mu_3y_5 - \frac{1}{6}\sigma_3y_6 - \frac{21}{64}\gamma_{16}y_6^3 - \frac{1}{12}\gamma_{24}y_1^2y_6 - \\ & \frac{1}{12}\gamma_{22}y_3^2y_6 - \frac{1}{12}\gamma_{22}y_4^2y_6 - \frac{21}{64}\gamma_{16}y_5^2y_6, \end{aligned} \quad (10e)$$

$$\begin{aligned} \dot{y}_6 = & \frac{1}{6}\sigma_3y_5 - \frac{1}{2}\mu_3y_6 + \frac{21}{64}\gamma_{16}y_5^3 + \frac{1}{12}\gamma_{24}y_1^2y_5 + \\ & \frac{1}{12}\gamma_{22}y_3^2y_5 + \frac{1}{12}\gamma_{22}y_4^2y_5 + \frac{21}{64}\gamma_{16}y_5y_6^2 - \frac{1}{6}F_3, \end{aligned} \quad (10f)$$

其中所用的非线性变换见附录。

为便于分析,将系统(10)的后四维方程转化成极坐标形式,即引入如下变换:

$$y_3 = I_1 \cos \theta_1, \quad y_4 = I_1 \sin \theta_1, \quad y_5 = I_2 \cos \theta_2, \quad y_6 = I_2 \sin \theta_2. \quad (11)$$

将变换(11)代入到三阶规范型(10)中得到如下的系统:

$$\dot{y}_1 = -\frac{1}{2}\mu_1y_1 + (1 - \bar{\sigma}_1)y_2, \quad (12a)$$

$$\dot{y}_2 = \bar{\sigma}_1y_1 - \frac{1}{2}\mu_1y_2 + \frac{9}{4}\alpha_{14}y_1^3 + \frac{1}{2}\alpha_{21}y_1I_1^2 + \frac{1}{2}\alpha_{25}y_1I_2^2, \quad (12b)$$

$$\dot{I}_1 = -\frac{1}{2}\mu_2I_1 - \frac{1}{4}F_2 \sin \theta_1, \quad (12c)$$

$$I_1 \dot{\theta}_1 = \frac{1}{4}\sigma_2I_1 + \frac{63}{128}\beta_{15}I_1^3 - \frac{1}{8}\beta_{20}y_1^2I_1 + \frac{1}{16}\beta_{23}I_1I_2^2 - \frac{1}{4}F_2 \cos \theta_1, \quad (12d)$$

$$\dot{I}_2 = -\frac{1}{2}\mu_3I_2 - \frac{1}{6}F_3 \sin \theta_2, \quad (12e)$$

$$I_2 \dot{\theta}_2 = \frac{1}{6}\sigma_3I_2 + \frac{21}{64}\gamma_{16}I_2^3 + \frac{1}{12}\gamma_{24}y_1^2I_2 + \frac{1}{12}\gamma_{22}I_2^2I_2 - \frac{1}{6}F_3 \cos \theta_2. \quad (12f)$$

为将系统(12)化为更简单的形式,引入如下变换:

$$\begin{pmatrix} y_1 \\ y_2 \\ I_1 \end{pmatrix} = \begin{pmatrix} \frac{\alpha_{25}}{\gamma_{25}}\sqrt{1 - \bar{\sigma}_1} & 0 & 0 \\ \alpha_{25} & \mu_1 & 1 \\ 2\gamma_{25}\sqrt{1 - \bar{\sigma}_1} & \sqrt{1 - \bar{\sigma}_1} & 0 \\ 0 & 0 & 1 \end{pmatrix} \begin{pmatrix} u_1 \\ u_2 \\ I_1 \end{pmatrix}, \quad (13)$$

其中  $\beta_{23}\gamma_{22} = -2\beta_{20}\gamma_{24}$ .

将变换(13)代入系统(12)中得到三自由度环形桁架天线系统的简单规范型:

$$\dot{u}_1 = c_1u_2, \quad (14a)$$

$$\dot{u}_2 = -c_2u_1 - \mu_1u_2 + c_3u_1^3 + c_4u_1I_1^2 + c_5u_1I_2^2, \quad (14b)$$

$$\dot{I}_1 = c_6I_1 - \frac{1}{4}F_2 \sin \theta_1, \quad (14c)$$

$$I_1 \dot{\theta}_1 = c_7I_1 + c_8I_1^3 + c_9u_1^2I_1 + c_{10}I_1I_2^2 - \frac{1}{4}F_2 \cos \theta_1, \quad (14d)$$

$$\dot{I}_2 = c_{11}I_2 - \frac{1}{6}F_3\sin\theta_2, \quad (14e)$$

$$I_2\dot{\theta}_2 = c_{12}I_2 + c_{13}I_2^3 + c_5u_1^2I_2 + c_{10}I_1^2I_2 - \frac{1}{6}F_3\cos\theta_2, \quad (14f)$$

其中

$$c_1 = \frac{\gamma_{25}}{\alpha_{25}}, c_2 = \frac{\alpha_{25}[1 - 4\bar{\sigma}_1(1 - \bar{\sigma}_1)]}{4\gamma_{25}}, c_3 = \frac{9\alpha_{14}\alpha_{25}^3(1 - \bar{\sigma}_1)^2}{4\gamma_{25}^3},$$

$$c_4 = \frac{\alpha_{21}\alpha_{25}}{2\gamma_{25}}(1 - \bar{\sigma}_1), c_5 = \frac{\alpha_{25}^2}{2\gamma_{25}}(1 - \bar{\sigma}_1), c_6 = -\frac{1}{2}\mu_2, c_7 = \frac{1}{4}\sigma_2, c_8 = \frac{63}{128}\beta_{15},$$

$$c_9 = \frac{1}{8}\beta_{20}\frac{\alpha_{25}^2}{\gamma_{25}}(1 - \bar{\sigma}_1), c_{10} = \frac{1}{16}\beta_{23}, c_{11} = -\frac{1}{2}\mu_3, c_{12} = \frac{1}{6}\sigma_3, c_{13} = \frac{21}{64}\gamma_{16}.$$

为研究阻尼参数对系统非线性动力学的影响,下面对阻尼参数引入扰动项  $\varepsilon$ :

$$\mu_1 \rightarrow \varepsilon\mu_1, \mu_2 \rightarrow \varepsilon\mu_2, \mu_3 \rightarrow \varepsilon\mu_3, F_2 \rightarrow \varepsilon F_2, F_3 \rightarrow \varepsilon F_3. \quad (15)$$

因此,系统(14)可以写成带有扰动项的 Hamilton 系统:

$$\dot{u}_1 = \frac{\partial H}{\partial u_2} + \varepsilon g^{u_1} = c_1u_2, \quad (16a)$$

$$\dot{u}_2 = -\frac{\partial H}{\partial u_1} + \varepsilon g^{u_2} = -c_2u_1 + c_3u_1^3 + c_4u_1I_1^2 + c_5u_1I_2^2 - \varepsilon\mu_1u_2, \quad (16b)$$

$$\dot{I}_1 = \frac{\partial H}{\partial \theta_1} + \varepsilon g^{I_1} = \varepsilon c_6I_1 - \frac{1}{4}\varepsilon F_2\sin\theta_1, \quad (16c)$$

$$I_1\dot{\theta}_1 = -\frac{\partial H}{\partial I_1} + \varepsilon g^{\theta_1} = c_7I_1 + c_8I_1^3 + c_9u_1^2I_1 + c_{10}I_1I_2^2 - \frac{1}{4}\varepsilon F_2\cos\theta_1, \quad (16d)$$

$$\dot{I}_2 = \frac{\partial H}{\partial \theta_2} + \varepsilon g^{I_2} = \varepsilon c_{11}I_2 - \frac{1}{6}\varepsilon F_3\sin\theta_2, \quad (16e)$$

$$I_2\dot{\theta}_2 = -\frac{\partial H}{\partial I_2} + \varepsilon g^{\theta_2} = c_{12}I_2 + c_{13}I_2^3 + c_5u_1^2I_2 + c_{10}I_1^2I_2 - \frac{1}{6}\varepsilon F_3\cos\theta_2. \quad (16f)$$

当  $\varepsilon = 0$  时,未扰系统(16)是完全可积的 Hamilton 系统,其对应的 Hamilton 函数为如下形式:

$$H = \frac{1}{2}c_1 + \frac{1}{2}c_2u_1^2 - \frac{1}{4}c_3u_1^4 - \frac{1}{2}(c_4I_1^2 + c_5I_2^2)u_1^2 - \frac{1}{2}c_7I_1^2 -$$

$$\frac{1}{4}c_8I_1^4 - \frac{1}{2}c_{10}I_1^2I_2^2 - \frac{1}{2}c_{12}I_2^2 - \frac{1}{4}c_{13}I_2^4. \quad (17)$$

当  $\varepsilon \neq 0$  时,系统(16)是耗散扰动系统,其耗散扰动项为如下形式:

$$g^{u_1} = 0, g^{u_2} = -\mu_1u_2, g^{I_1} = c_6I_1 - \frac{1}{4}F_2\sin\theta_1, g^{\theta_1} = -\frac{1}{4}F_2\cos\theta_1,$$

$$g^{I_2} = c_{11}I_2 - \frac{1}{6}F_3\sin\theta_2, g^{\theta_2} = -\frac{1}{6}F_3\cos\theta_2. \quad (18)$$

## 2 未扰系统动力学分析

首先考虑系统未受扰动的情形,即当  $\varepsilon = 0$  时,系统(16)是一个解耦的两自由度未扰动非线性系统.当  $\varepsilon = 0$  时,变量  $\dot{I}_1 = 0, \dot{I}_2 = 0$ .因此,变量  $I_1$  和  $I_2$  为常数,在系统(16)的子空间  $(u_1,$

$u_2$ ) 中可以看作为参数,系统(16)的前两个方程与后面四个方程是解耦的,下面先研究系统(16)在未扰情形下的前两个方程:

$$\dot{u}_1 = c_1 u_2, \quad (19a)$$

$$\dot{u}_2 = -c_2 u_1 + c_3 u_1^3 + c_4 u_1 I_1^2 + c_5 u_1 I_2^2. \quad (19b)$$

当  $c_3 c_4 < 0$  时,系统(16)会出现同宿分叉.在此令  $c_3 > 0, c_4 < 0$ .当  $c_2 - c_4 I_1^2 - c_5 I_2^2 < 0$  时,系统(16)只有一个初始平衡点  $(u_1, u_2) = (0, 0)$ .通过计算系统(19)在初始平衡点  $(u_1, u_2) = (0, 0)$  处 Jacobi 矩阵可知初始平衡点为中心点,接下来定义临界曲线.令  $c_2 - c_4 I_1^2 - c_5 I_2^2 = 0$  得到初始平衡点  $(u_1, u_2) = (0, 0)$  在原点处发生 pitchfork 分叉产生三个解,鞍点  $P_1 = (0, 0)$  和中心点  $P_{2,3} = (\pm B, 0)$ , 其中

$$B = \left[ \frac{1}{c_3} (c_2 - c_4 I_1^2 - c_5 I_2^2) \right]^{1/2}. \quad (20)$$

在系统中,变量  $I_1, I_2$  和  $\theta_1, \theta_2$  分别代表系统振动的幅值和相位,所以,变量  $I_1$  和  $I_2$  应该满足  $I_i \geq 0, i = 1, 2$ , 在  $I \in (I_1, I_2) \notin [I_{11}, I_{12}] \times [I_{21}, I_{22}] \subset R^2$  中,未扰系统(19)有两个中心点  $P_{2,3} = (\pm B, 0)$ 、一个鞍点  $P_1 = (0, 0)$  以及一对同宿轨道  $u_{\pm}^h(T_1, I_i), i = 1, 2$ , 且同宿轨道满足  $\lim_{T_1 \rightarrow \pm\infty} u_{\pm}^h(T_1, I_i) = P_1, i = 1, 2$ , 其中  $T_1$  是定义在二维扰动慢变流形上的相空间上.

未扰系统(19)在相空间  $(u_1, u_2)$  上不同区域内平衡点的相图如图 2 所示.当  $c_2 - c_4 I_1^2 - c_5 I_2^2 < 0$  时,系统只有一个平衡点且为中心点;当  $c_2 - c_4 I_1^2 - c_5 I_2^2 > 0$  时,系统有三个平衡点,其中一个鞍点两个中心点且连接鞍点的是一对同宿轨道,而此同宿轨道亦为中心点的外边界.

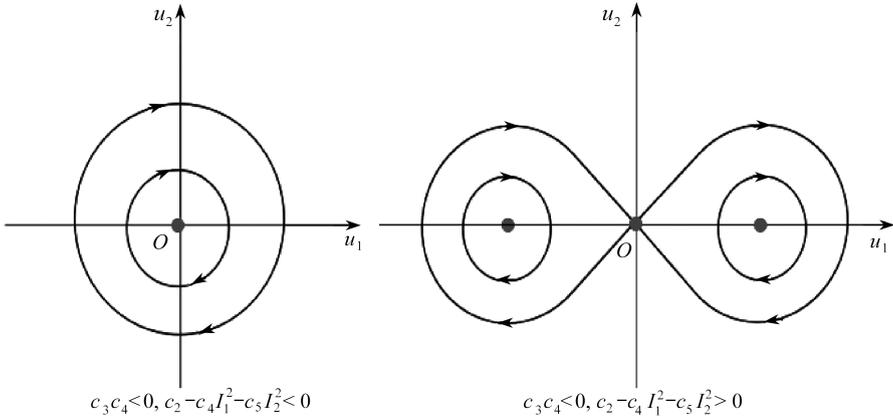


图 2 未扰系统平衡点的相图

Fig. 2 The phase portrait of the equilibrium points of the unperturbed system

在六维相空间中四维正规双曲不变流形定义为如下形式:

$$M = \{ (u, I, \theta) \in R^2 \times R^2 \times S^2 \mid u = P_1, I_1 < I < I_2, 0 \leq \theta_i < 2\pi \}, \quad (21)$$

$i = 1, 2.$

不变流形  $M$  有五维稳定流形和不稳定流形分别为  $W^s(M)$  和  $W^u(M)$ .在系统(21)中由同宿轨道连接的奇点沿五维稳定流形和不稳定流形形成如下五维的同宿流形  $\Gamma$ .其中由连接奇点  $P_1 = (0, 0)$  的同宿轨道可知  $W^s(M)$  和  $W^u(M)$  沿五维同宿流形  $\Gamma$  非横截相交,同宿流形  $\Gamma$  定义为如下形式:

$$\Gamma = \left\{ (u, I, \theta) \mid u = u_{\pm}^h(T_1, I_i), I \in (I_1, I_2), \gamma = \int_0^{T_1} \frac{\partial H(u_{\pm}^h(T_1, I), I)}{\partial I_i} ds + \theta_{i0} \right\}, \quad (22)$$

$i = 1, 2.$

对于系统振动的每个幅值  $I \in (I_1, I_2)$  时, 二维正规双曲不变圆环  $\theta(I)$  具有三维稳定流形  $W^s(\theta(I))$  和不稳定流形  $W^u(\theta(I))$  重合形成的三维同宿流形, 其中六维相空间中流形的结构图如图 3 所示,  $\gamma$  表示振动相位.

未扰系统(19)的前两维系统定义在二维扰动慢变流形动的相空间上. 下面分析限制在流形  $M$  上的未扰系统(19)为

$$\dot{I}_1 = 0, \tag{23a}$$

$$I_1 \dot{\theta}_1 = c_7 I_1 + c_8 I_1^3 + c_9 u_1^2 I_1 + c_{10} I_1 I_2^2, \tag{23b}$$

$$\dot{I}_2 = 0, \tag{23c}$$

$$I_2 \dot{\theta}_2 = c_{12} I_2 + c_{13} I_2^3 + c_5 u_1^2 I_2 + c_{10} I_1^2 I_2. \tag{23d}$$

当  $I \in (I_1, I_2) = C$ ,  $\frac{\partial H(q_{\pm}(I), I)}{\partial I_i} \neq 0, i = 1, 2$  时, 系统的解为含有两个参数的二维圆环;

当  $I \in (I_1, I_2) = C$ ,  $\frac{\partial H(q_{\pm}(I), I)}{\partial I_i} = 0, i = 1, 2$  时, 系统的解为不动点圆环, 其中  $\frac{\partial H(q_{\pm}(I), I)}{\partial I_i}$ ,

$i = 1, 2$  的表达式如下:

$$\frac{\partial H(q_{\pm}(I), I)}{\partial I_1} = c_7 I_{1r} + c_8 I_{1r}^3 + c_9 q_{\pm}^2 I_{1r} + c_{10} I_{1r} I_{2r}^2, \tag{24a}$$

$$\frac{\partial H(q_{\pm}(I), I)}{\partial I_2} = c_{12} I_{2r} + c_{13} I_{2r}^3 + c_5 q_{\pm}^2 I_{2r} + c_{10} I_{1r}^2 I_{2r}. \tag{24b}$$

此时使得等式  $\frac{\partial H(q_{\pm}(I), I)}{\partial I_i} = 0, i = 1, 2$  成立的  $I$  值为系统的共振值, 记为  $I_r = (I_{1r}, I_{2r})$ . 则可

得到如下的共振值:

$$I_{1r}^2 = \frac{c_{10} c_{12} - c_7 c_{13}}{c_8 c_{13} - c_{10}^2}, \tag{25a}$$

$$I_{2r}^2 = \frac{c_7 c_{10} - c_8 c_{12}}{c_8 c_{13} - c_{10}^2}. \tag{25b}$$

为计算系统(19)连接双曲鞍点  $P_1 = (0, 0)$  的同宿轨道以及相位差  $\Delta\theta$ , 给出未扰系统(19)未扰动部分的 Hamilton 函数:

$$\hat{H} = \frac{1}{2} c_1 u_2^2 - \frac{1}{4} c_3 u_1^4 - \frac{1}{2} (c_4 I_1^2 + c_5 I_2^2 - c_2) u_1^2. \tag{26}$$

令  $\varepsilon_1 = c_4 I_1^2 + c_5 I_2^2 - c_2$ , 联立鞍点  $P_1 = (0, 0)$  和 Hamilton 函数并化简得到系统(19)的同宿轨道:

$$u_1(T_1) = \pm \sqrt{\frac{2\varepsilon_1}{-c_3}} \operatorname{sech}(\sqrt{c_1 \varepsilon_1} T_1), \tag{27a}$$

$$u_2(T_1) = \mp \varepsilon_1 \sqrt{\frac{2}{-c_3}} \tanh(\sqrt{c_1 \varepsilon_1} T_1) \operatorname{sech}(\sqrt{c_1 \varepsilon_1} T_1). \tag{27b}$$

将轨道(27)代入式(20)中并对式(23b)和(23d)进行积分, 得到系统的振动相位  $\theta_1$  和  $\theta_2$  为如下形式:

$$\theta_1 = e_1 T_1 + \frac{2c_9 \sqrt{e}}{c_3 \sqrt{c_1}} \tanh(\sqrt{c_1 e} T_1) + \theta_{01}, \tag{28a}$$

$$\theta_2 = e_2 T_1 + \frac{2c_5 \sqrt{e}}{c_3 \sqrt{c_1}} \tanh(\sqrt{c_1} e T_1) + \theta_{02}, \quad (28b)$$

其中  $e = c_2 - c_4 I_1^2 - c_5 I_2^2, e_1 = c_7 + c_8 I_1^2 + c_{10} I_2^2, e_2 = c_{12} + c_{10} I_1^2 + c_{13} I_2^2, \theta_{01}$  和  $\theta_{02}$  为初始相位角。

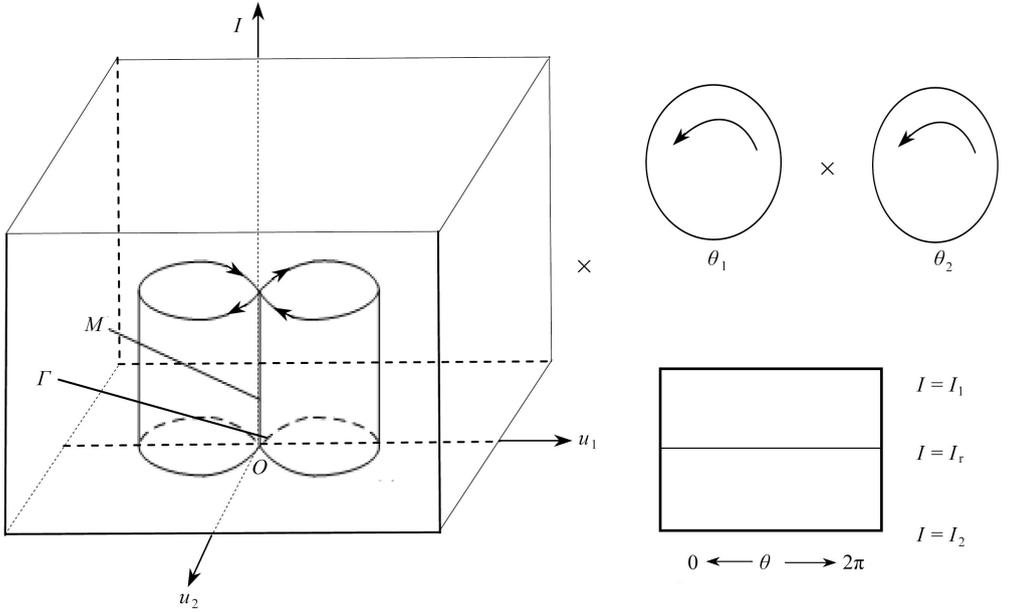


图3 六维空间中不变流形  $M$  与同宿流形  $\Gamma$  的结构图

Fig. 3 The structure portrait of invariant manifold  $M$  and homoclinic manifold  $\Gamma$  in the 6D space

在此定义相位差为如下形式：

$$\Delta\theta_i = \theta(+\infty, I_i) - \theta(-\infty, I_i), \quad i = 1, 2. \quad (29)$$

当  $I_i = I_{ir}(i = 1, 2)$  时, 即共振情况下,  $d_i \equiv 0, i = 1, 2$ , 则相位差为

$$\Delta\theta_1 = \frac{4c_9 \sqrt{\varepsilon_1}}{c_3 \sqrt{c_1}}, \quad \Delta\theta_2 = \frac{4c_5 \sqrt{\varepsilon_1}}{c_3 \sqrt{c_1}}. \quad (30)$$

### 3 扰动系统动力学分析

综上所述, 得到了未扰系统(19)的同宿轨道、相位差等, 并分析了在六维相空间中流形  $M$  的几何结构. 接下来将研究系统(19)在小扰动影响下的动力学特性. 根据文献[16-17]可知, 在小扰动下不变流形  $M$  沿稳定流形与不稳定流形是不变的,  $P_1$  仍为鞍点且此时的流形  $M_\varepsilon$  充分接近  $M$ , 即定义

$$M = M_\varepsilon = \{(u_1, u_2, I, \theta) \mid (u_1, u_2) = P_1, I_{i1} < I < I_{i2}, 0 \leq \theta_i < 2\pi\}, \quad i = 1, 2. \quad (31)$$

因此将系统(16)限制在流形  $M_\varepsilon$  上, 在共振情形下研究系统(16)的动力学特性. 首先定义一个四维空间  $M_4 = (u_1, u_2, I_1, \theta_1)$  为如下形式:

$$M_4 = [(u_1, u_2, I_1, \theta_1) \in R \times R \times R \times S \mid (u_1, u_2) = P_1, I_1 \in (I_{11}, I_{12}), 0 < \theta_1 \leq 2\pi]. \quad (32)$$

将系统中的变量  $I_2$  限制在  $[I_{2r} - \varepsilon, I_{2r} + \varepsilon] \subset [I_{21}, I_{22}]$  上, 且变量  $\theta_2$  满足  $\theta_2 \in (0, 2\pi]$ . 只需研究在共振情形下系统  $M_4 = (u_1, u_2, I_1, \theta_1)$  的动力学特性, 之后让变量  $I_2$  取遍  $[I_{2r} - \varepsilon, I_{2r}$

$+\varepsilon] \subset [I_{21}, I_{22}]$  上的每个点, 从而就可以到整个六维系统(16)的动力学特性.

为得到系统未扰动部分 Hamilton 函数, 引入如下变换:

$$I_1 \rightarrow I_{1r} + \sqrt{\varepsilon}h, \quad T_1 \rightarrow \sqrt{\varepsilon}T_1. \quad (33)$$

则将变换(33)代入式(16c)和(16d)并化简得到

$$\dot{h} = c_6 I_{1r} - \frac{1}{4} F_2 \sin \theta_1 + c_6 \sqrt{\varepsilon}h, \quad (34a)$$

$$\dot{\theta}_1 = 2I_{1r}c_8h + \sqrt{\varepsilon}c_8h^2 - \frac{\sqrt{\varepsilon}F_2}{4I_{1r}} \cos \theta_1. \quad (34b)$$

未扰系统(34)是 Hamilton 系统, 其对应的 Hamilton 函数为如下形式:

$$\Delta \hat{H}_D(h, \theta_1) = c_6 I_{1r} \theta_1 - c_8 I_{1r} h^2 + \frac{1}{4} F_2 \cos \theta_1. \quad (35)$$

当  $\varepsilon = 0$ , 在  $\theta_1 \in (0, 2\pi)$  区间内得到系统(34)的不动点为

$$Q_1 = (0, \theta_{1s}) = \left(0, \pi - \arcsin \frac{4c_6 I_{1r}}{F_2}\right), \quad Q_2 = (0, \theta_{1c}) = \left(0, \arcsin \frac{4c_6 I_{1r}}{F_2}\right). \quad (36)$$

下面计算在奇点  $Q_1$  和  $Q_2$  处的 Jacobi 矩阵, 分析奇点的稳定性, 则系统(34)未扰动部分的 Jacobi 矩阵为

$$J = \begin{bmatrix} 0 & -\frac{1}{4} F_2 \cos \theta_1 \\ 2c_8 I_{1r} & 0 \end{bmatrix}. \quad (37)$$

根据方程(37), 当  $(2\delta \bar{f}_2 I_r \cos \theta_c) / \beta_1 > 0$  时, 系统有一对纯虚特征根, 即奇点  $Q_2$  为中心点; 当  $(2\delta \bar{f}_2 I_r \cos \theta_c) / \beta_1 < 0$  时, 系统(34)有两个异号的实特征根, 即奇点  $Q_1$  为鞍点且由一对同宿轨道连接.

在小扰动  $\varepsilon$  下, 奇点  $Q_1$  仍为双曲奇点, 扰动后的奇点  $Q_{\varepsilon_1}$  仍为鞍点. 而奇点  $Q_2$  在受扰动后将会产生变化, 首先考虑系统(34)线性部分的 Jacobi 矩阵:

$$J_{Q_{2\varepsilon}} = \begin{bmatrix} c_6 \sqrt{\varepsilon} & -\frac{1}{4} F_2 \cos \theta_1 \\ 2c_8 I_{1r} & \frac{\sqrt{\varepsilon} F_2}{4I_{1r}} \sin \theta_1 \end{bmatrix}. \quad (38)$$

根据矩阵(38), 我们发现系统(37)线性部分的低阶项在同宿环内小于零. 因此, 在扰动下奇点  $Q_2$  变为双曲焦点  $Q_{\varepsilon_2}$ .

## 4 能量相位法

在本节利用 Haller 和 Wiggins<sup>[16-17]</sup> 提出的能量相位法以及改进后的能量相位法, 研究环形桁架天线具有 Shilnikov 同宿轨道的多脉冲混沌运动. 计算相空间中快变流行上每个脉冲跳跃所消耗的能量, 即能量差分函数的具体表达式以及能量差分函数的横截零点, 从而验证系统存在着多脉冲轨道.

首先给出系统(16)的含有耗散项的能量差分函数:

$$\Delta^N \hat{H}_D(u_1, u_2, I_1, I_2, \theta_1, \theta_2) = \hat{H}_D(h, \theta_1 + N\Delta\theta_1) - \hat{H}_D(h, \theta_1) - n \int_A \left[ \frac{d}{du_1} g^{u_1}(u_1, u_2, I, \theta) \frac{d}{du_2} g^{u_2}(u_1, u_2, I, \theta) \right] du_1 du_2 - n \int_{\partial A} g^I d\theta, \quad (39)$$

其中  $\Delta\theta$  是相位差,  $A$  为在子空间  $(u_1, u_2)$  中由一对同宿轨道围成的区域,  $\partial A_l$  是区域  $A$  的边界. 由式(39)可知能量差分函数分为四个部分:

$\hat{H}_D(h, \theta_1 + N\Delta\theta_1)$  为  $n$  个脉冲的总能量;  $\Delta\hat{H}_D(h, \theta_1)$  为在  $(h, \theta)$  平面中未跳起脉冲的能量, 即

$$\hat{H}_D(h, \theta_1 + N\Delta\theta_1) - \hat{H}_D(h, \theta_1) = c_6 I_{1r} \Delta\theta_1 + \frac{1}{4} F_2 (\cos(\theta_1 + n\Delta\theta_1) - \cos\theta_1); \quad (40)$$

$n \int_A \left[ \frac{d}{du_1} g^{u_1}(u_1, u_2, I_r, \theta_r) + \frac{d}{du_2} g^{u_2}(u_1, u_2, I_r, \theta_r) \right] du_1 du_2$  为同宿轨道所围成区域的面积, 即

$$\begin{aligned} n \int_A \left[ \frac{d}{du_1} g^{u_1}(u_1, u_2, I_r, \theta_r) + \frac{d}{du_2} g^{u_2}(u_1, u_2, I_r, \theta_r) \right] du_1 du_2 = \\ - \frac{4n\mu_1 \varepsilon_1 \Delta\theta_1}{3c_5 \sqrt{c_1}} - nc_{11} I_{2r} \Delta\theta_2; \end{aligned} \quad (41)$$

$n \int_{\partial A_l} g^l d\theta$  为脉冲沿着同宿轨道的积分, 即

$$n \int_{\partial A_l} g^l d\theta = -nc_6 I_{1r} \Delta\theta_1 - nc_{11} I_{2r} \Delta\theta_2. \quad (42)$$

经计算得到系统(16)的能量差分函数的具体表达式:

$$\begin{aligned} \Delta^N \hat{H}_D(u_1, u_2, I_1, I_2, \theta_1, \theta_2) = \\ - \frac{1}{2} F_2 \sin\left(\frac{n}{2} \Delta\theta_1 + \theta_1\right) \sin\left(\frac{n}{2} \Delta\theta_1\right) + \frac{4n\mu_1 \varepsilon_1 \Delta\theta_1}{3c_5 \sqrt{c_1}} + nc_{11} I_{2r} \Delta\theta_2. \end{aligned} \quad (43)$$

接下来计算能量差分函数的横截零点. 首先, 定义一个包含能量差分函数所有横截零点的集合:

$$\begin{aligned} \hat{Z}_-^N = \left\{ (h, \theta) \mid \Delta^N \hat{H}_D(u_1, u_2, I_1, I_2, \theta_1, \theta_2) = 0, \right. \\ \left. \frac{\partial \Delta^N \hat{H}_D(u_1, u_2, I_1, I_2, \theta_1, \theta_2)}{\partial \theta} \neq 0 \right\}. \end{aligned} \quad (44)$$

含耗散项的能量差分函数  $\Delta^N \hat{H}_D(u_1, u_2, I_1, I_2, \theta_1, \theta_2)$  的横截零点满足如下条件:

$$\theta_1 + \frac{n\Delta\theta_1}{2} = 2m\pi + (-1)^m \alpha, \quad (45)$$

其中  $m \in \mathbf{Z}$ , 并且得到

$$\alpha = \arcsin \left[ \frac{8n\mu_1 \varepsilon_1 \Delta\theta_1 + 6nc_5 \sqrt{c_1} c_{11} I_{2r} \Delta\theta_2}{3F_2 c_5 \sqrt{c_1} \sin((n/2) \cdot \Delta\theta_1)} \right]. \quad (46)$$

在区间  $\theta_1 \in \left[ -\frac{\pi}{2}, \frac{3\pi}{2} \right]$  上, 当含有耗散项的能量差分函数  $\Delta^N \hat{H}_D(u_1, u_2, I_1, I_2, \theta_1, \theta_2)$  满足  $n\Delta\theta_1 \neq 4l\pi (l = 0, 1, 2, \dots)$  时, 能量差分函数有如下两个横截零点:

$$\theta_{0,1}^n = \frac{3\pi}{2} - \left( \frac{n\Delta\theta_1}{2} + \alpha \right) \bmod(2\pi), \quad (47a)$$

$$\theta_{0,2}^n = \frac{3\pi}{2} - \left( \pi + \frac{n\Delta\theta_1}{2} - \alpha \right) \bmod(2\pi). \quad (47b)$$

由集合(43)可以得出如下关系式:

$$\sin\left(\frac{n}{2}\Delta\theta_1 + \theta_1\right) = \frac{8n\mu_1\varepsilon_1\Delta\theta_1 + 6nc_5\sqrt{c_1}c_{11}I_{2r}\Delta\theta_2}{3F_2c_5\sqrt{c_1}\sin\left(\frac{n}{2}\Delta\theta_1\right)}. \quad (48)$$

为研究系统阻尼与激励之间的关系,定义变量  $d = \mu/F_2$  为耗散因子,令  $\mu = \mu_1 = \mu_2 = \mu_3$ , 则由方程(48)得到耗散因子的上确界:

$$|d| < d_{\max} = \left| \frac{3c_5\sqrt{c_1}\sin((n/2)\cdot\Delta\theta_1)}{8n\varepsilon_1\Delta\theta_1 - 3nc_5\sqrt{c_1}I_{2r}\Delta\theta_2} \right|. \quad (49)$$

同理,可得到脉冲数的上确界

$$n < n_{\max} = \left\lfloor \frac{3c_5\sqrt{c_1}}{8d\varepsilon_1\Delta\theta_1 - 3dc_5\sqrt{c_1}I_{2r}\Delta\theta_2} \right\rfloor. \quad (50)$$

为分析环形桁架天线系统是否存在 Shilnikov 型多脉冲轨道,需要验证如下 3 个条件是否成立:① 系统的 Hamilton 函数  $\Delta\hat{H}_D(h, \theta_1)$  存在非退化零点;② 在整个六维相空间  $(u_1, u_2, I_1, I_2, \theta_1, \theta_2)$  中,非退化零点是耗散能量差分函数的横截零点且满足非退化条件;③ 从慢流形上跳起的脉冲轨道最后又落回焦点的吸引域内。

首先由方程(35)得到了 Hamilton 函数  $\Delta\hat{H}_D(h, \theta_1)$  的非退化零点为  $Q_2$ 。

接下来将中心点  $Q_2$  代入能量差分函数中并计算其零点,即

$$\Delta^N\hat{H}_D(u_1, u_2, I_1, I_2, \theta_1, \theta_2) \Big|_{Q_2} = 0. \quad (51)$$

化简后得到耗散因子的表达式:

$$d = \frac{3c_5\sqrt{c_1}[\sqrt{F_2^2 - 16c_6^2I_{1r}^2}(1 - \cos(n\Delta\theta_1))] + 4c_6I_{1r}\sin(n\Delta\theta_1)}{16n\varepsilon_1\Delta\theta_1F_2 - 6c_5\sqrt{c_1}I_{2r}\Delta\theta_2F_2}. \quad (52)$$

当耗散因子不为零时,即如下条件成立时,中心点  $Q_2$  为能量差分函数零点:

$$n\Delta\theta_1 \neq 2k\pi, \quad k \in \mathbf{Z}_+, \quad (53)$$

$$\frac{\partial\Delta\hat{H}_D(h, \theta_1)}{\partial d} = \frac{1}{4}F_2(\cos(\theta_1 + n\Delta\theta_1) - \cos\theta_1) + \frac{4n\mu_1\varepsilon_1\Delta\theta_1}{3c_5\sqrt{c_1}} + nc_{11}I_{2r}\Delta\theta_2. \quad (54)$$

经验证,在条件(53)情形下,式(51)与(54)不能同时成立,即中心点  $Q_2$  是能量差分函数的横截零点。

最后,只需验证从慢流形上跳起的脉冲最终又落回到慢流形上中心点  $Q_2$  的吸引域内。

在慢流形上,我们知道脉冲的起跳点为中心点  $Q_2$ , 其对应的能量函数为  $\hat{H}_D(0, \theta_{1c})$ , 鞍点对应的能量为  $\hat{H}_D(0, \theta_{1s})$ . 在区间  $(0, 2\pi)$  上,假设脉冲的落回点为  $Q$ , 其对应的能量函数为  $\hat{H}_D(0, \theta_{1*})$ , 由于相图的对称性,当落回点与鞍点或中心点距离超过鞍点对应的能量为  $2k\pi$  时,重新定义落回点为如下形式:

$$\theta_{1*}^N = \theta_{1s} + (\theta_{1c} + n\Delta\theta_1 - \theta_{1s}) \bmod(2k\pi), \quad (55)$$

其中

$$\theta_{1s} = \pi + \arcsin(2dI_{1r}), \quad \theta_{1c} = -\arcsin(2dI_{1r}). \quad (56)$$

起跳点处的能量最大,当落回点不断远离起跳点时,能量不断减小,能量最小的落回点为鞍点,当落回点的能量满足如下条件时,落回点就落到了焦点的吸引域内:

$$\hat{H}_D(0, \theta^*) > \hat{H}_D(0, \theta_s). \quad (57)$$

下面以未扰系统的鞍点为吸引域的边界,在区间  $(0, 4\pi)$  内定义两个区域:

$$D_1 = (\pi + \arcsin(2dI_{1r}), 3\pi + \arcsin(2dI_{1r})), \quad (58a)$$

$$D_2 = (3\pi + \arcsin(2dI_{lr}), 5\pi + \arcsin(2dI_{lr})) . \quad (58b)$$

则落回点会落到两个区域中的一个或两个区域平移  $2k\pi$  后的区域内.如果落回点落入区域  $D_1$  内,则存在连接鞍点自身的 Shilnikov 型同宿轨道;如果落回点落入区域  $D_2$  内,则存在连接两个鞍点的异宿轨道.

## 5 数值模拟

为了验证上述理论分析,对系统的平均方程(6)进行数值模拟.当系统(6)中的参数  $c_3c_4 = -\frac{9\alpha_{14}\alpha_{25}^2}{4\gamma_{25}^2} < 0$ , 即  $\alpha_{14} > 0$  时,系统会出现 Shilnikov 型多脉冲混沌运动.在上述定性分析的基础上发现激励参数在环形桁架天线多脉冲混沌运动中起着重要作用.

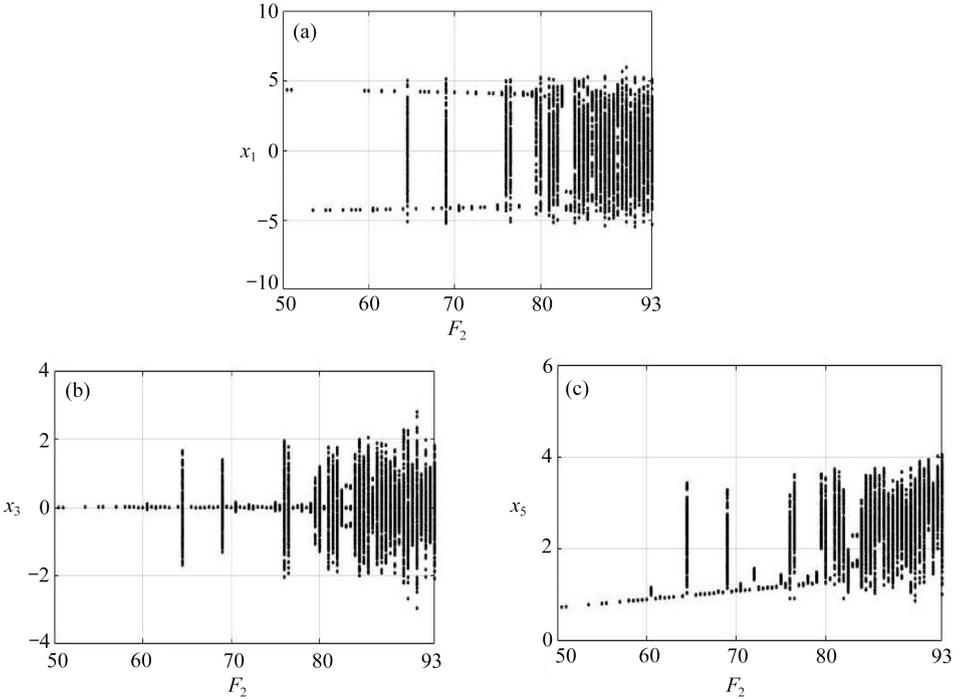
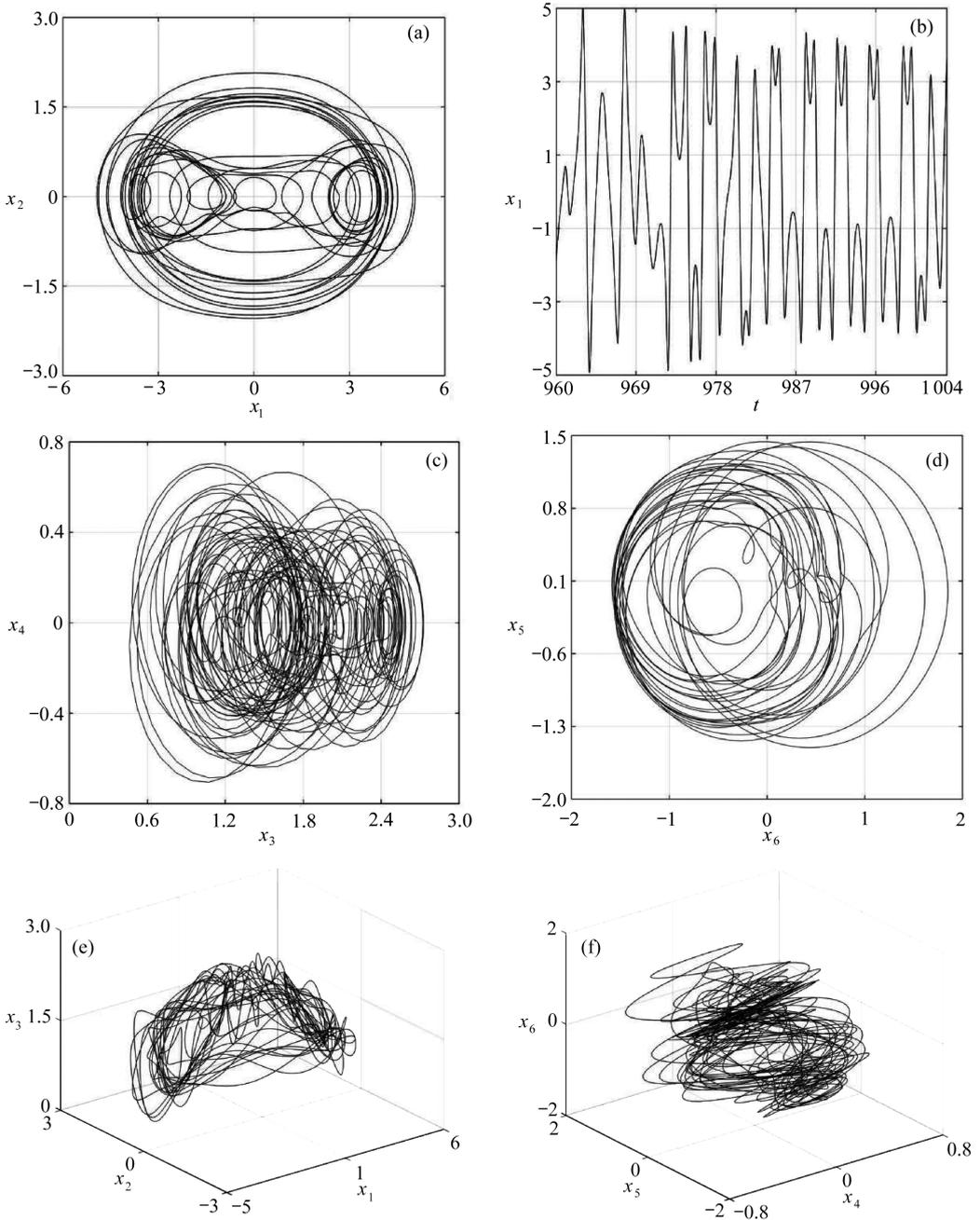


图4 系统随激励  $F_2$  变化的分叉图

Fig. 4 The bifurcation diagram of the system under excitation  $F_2$

选择激励参数  $F_2$  作为系统的控制参数,研究等效圆柱壳的非线性动力学行为.选取系统的初值为  $x_{10} = 0.04, x_{20} = 0.15, x_{30} = 0.35, x_{40} = 0.19, x_{50} = 0.01, x_{60} = 0.02$  以及系统的参数值为  $\mu_1 = 0.05, \mu_2 = 0.05, \mu_3 = 0.05, \sigma_1 = 0.12, \sigma_2 = 0.19, \sigma_3 = 3.12, \alpha_{14} = 0.11, \alpha_{21} = 0.9, \alpha_{25} = 3.9, \beta_{15} = 2.11, \beta_{20} = 7, \beta_{23} = 24, \beta_{24} = 2, \gamma_{16} = 2, \gamma_{20} = 5, \gamma_{22} = 2.5, \gamma_{24} = 2.3, f_1 = 5, F_3 = 17.5$ , 激励  $F_2$  在  $F_2 \in (50, 90)$  区间内环形桁架天线的分叉图如图4所示.从图4中可以看出,当激励  $F_2 < 80$  时,系统一直处于周期运动状态;当激励  $F_2 > 80$  时,系统进入混沌运动状态.选取激励  $F_2 = 92.1$ , 得到系统的混沌运动相图,如图5所示.其中图5(a)表示系统在  $(x_1, x_2)$  平面上的相图,图5(b)表示系统在  $(t, x_1)$  平面上的相图,图5(c)表示系统在  $(x_3, x_4)$  平面上的波形图,图5(d)表示系统在  $(x_6, x_5)$  平面上的波形图,图5(e)表示系统在  $(x_1, x_2, x_3)$  平面上的相图,图5(f)表示系统在  $(x_4, x_5, x_6)$  平面上的相图.从图5中发现系统的混沌运动轨迹有明显的跳跃现象.

图5  $F_2 = 92.1$  时系统的混沌运动图Fig. 5 The chaos motion diagram of the system for  $F_2 = 92.1$ 

## 6 结 论

本文首次研究了六维系统环形桁架天线 Shilnikov 型多脉冲混沌运动.选取横向振动为主要振动进行三阶离散,得到环形桁架天线呼吸振动的三自由度非线性方程.利用多尺度法得到 1:4:6 内共振关系下环形桁架天线的平均方程.利用规范型理论化简平均方程,根据推广后的能量相位法分析了其拓扑等价方程的非线性动力学行为.通过分析环形桁架天线多脉冲同宿

轨道,发现未扰系统中的鞍点存在一条连接自身的同宿轨道,当受到扰动后同宿轨破裂,其中一个分支落入焦点的吸引域内,另一个分支形成了焦点附近吸引域的边界,且焦点产生跳跃的脉冲轨道,最终又回到吸引域内.分析还表明,同宿轨道的存在依赖于能量相位判据.在数值模拟中发现激励参数对天线非线性运动影响较大.当选择一定的参数值和初始条件时,环形桁架天线会出现大幅度的振动.在一定区间内,系统保持稳定的周期运动.随着激励参数的增大,系统进入不稳定的混沌运动状态.本文从多个方面解释和描述了环形桁架天线可以产生 Shilnikov 型多脉冲混沌运动.综上理论分析与数值模拟,可以为抑制环形桁架天线产生混沌运动或避免大的振动提供参考.

## 附 录

$$x_1 = -\frac{3}{16}\alpha_{14}y_1^3 - \frac{9}{8}\alpha_{14}y_1y_2^2 - \frac{1}{4}\alpha_{21}y_1y_3^2 - \frac{1}{4}\alpha_{21}y_1y_4^2 - \frac{1}{4}\alpha_{25}y_1y_5^2 - \frac{1}{4}\alpha_{25}y_1y_6^2, \quad (A1)$$

$$x_2 = \frac{9}{16}\alpha_{14}y_1^2y_2, \quad (A2)$$

$$x_3 = -\frac{9\beta_{15}}{32\sigma_2}y_3^3 + \frac{9\beta_{15}}{32\sigma_2}y_4^3 - \frac{3\beta_{24}}{4(3\sigma_2 - 2\sigma_3)}y_1^2y_5 + \frac{3\beta_{24}y_2^2y_5}{4}[(81\sigma_2^4 - 72\sigma_2^2\sigma_3^2 + 16\sigma_3^4 - 129\sigma_2^3 + 864\sigma_2^2\sigma_3 - 576\sigma_2\sigma_3^2 + 384\sigma_3^3 + 10\ 368\sigma_2^2 + 13\ 824\sigma_2\sigma_3 + 4\ 608\sigma_3^2)/(234\sigma_2^5 - 162\sigma_2^4\sigma_3 - 216\sigma_2^3\sigma_3^2 + 144\sigma_2^2\sigma_3^3 - 162\sigma_2^4\sigma_3 - 216\sigma_2^2\sigma_3^2 + 144\sigma_2^2\sigma_3^3 + 48\sigma_2\sigma_3^4 - 32\sigma_3^5)] + \frac{8\beta_{24}y_2y_3^2}{9\sigma_2^2} + \frac{9\beta_{15}y_3^2y_4}{16\sigma_2} - \frac{8\beta_{24}y_2y_4^2}{9\sigma_2^2} - \frac{9\beta_{15}y_3y_4^2}{16\sigma_2} + \frac{\beta_{23}y_3y_5^2}{4\sigma_2} - \frac{\beta_{23}y_4y_5^2}{4\sigma_2} + \frac{\beta_{23}y_3y_6^2}{4\sigma_2} - \frac{\beta_{23}y_4y_6^2}{4\sigma_2} - \frac{\beta_{20}y_1y_2y_4}{16} + \frac{\beta_{24}y_1y_3y_4}{6\sigma_2} + \frac{3(3\sigma_2\sigma_3 - 2\sigma_3^2 - 36\sigma_2 - 24\sigma_3)\beta_{24}y_1y_2y_6}{27\sigma_2^3 - 18\sigma_2^2\sigma_3 - 12\sigma_2\sigma_3^2 + 8\sigma_3^3}, \quad (A3)$$

$$x_4 = -\frac{27\beta_{15}}{32\sigma_2}y_3^3 - \frac{27\beta_{15}}{32\sigma_2}y_4^3 - \frac{3\beta_{24}}{4(3\sigma_2 - 2\sigma_3)}y_1^2y_6 + \frac{3\beta_{24}y_2^2y_6}{4}[(81\sigma_2^4 - 72\sigma_2^2\sigma_3^2 + 16\sigma_3^4 - 1\ 728\sigma_2^2\sigma_3 + 1\ 152\sigma_2\sigma_3^2 + 10\ 368\sigma_2^2 + 13\ 824\sigma_2\sigma_3 + 4\ 608\sigma_3^2)/(243\sigma_2^5 - 162\sigma_2^4\sigma_3 - 216\sigma_2^3\sigma_3^2 + 144\sigma_2^2\sigma_3^3 + 48\sigma_2\sigma_3^4 - 32\sigma_3^5)] - \frac{\beta_{24}y_1y_3^2}{6\sigma_2} + \frac{\beta_{24}y_1y_4^2}{6\sigma_2} + \frac{\beta_{20}y_1y_2y_3}{16} - \frac{9(3\sigma_2^2 - 2\sigma_2\sigma_3 - 24\sigma_2 - 16\sigma_3)\beta_{24}y_1y_2y_5}{2(27\sigma_2^3 - 18\sigma_2^2\sigma_3 - 12\sigma_2\sigma_3^2 + 8\sigma_3^3)} + \frac{20\beta_{24}y_2y_3y_4}{9\sigma_2^2}, \quad (A4)$$

$$x_5 = -\frac{9\gamma_{16}}{32\sigma_3}y_5^3 + \frac{\gamma_{20}}{2(3\sigma_2 - 2\sigma_3)}y_1^2y_3 + \frac{\gamma_{20}}{96}y_1^2y_6 - \frac{\gamma_{20}y_2^2y_3}{2}[(81\sigma_2^4 - 72\sigma_2^2\sigma_3^2 + 16\sigma_3^4 - 864\sigma_2^2\sigma_3 + 576\sigma_2\sigma_3^2 - 384\sigma_3^3 + 10\ 368\sigma_2^2 + 13\ 824\sigma_2\sigma_3 + 1\ 296\sigma_2^3 + 4\ 608\sigma_3^2)/(243\sigma_2^5 - 162\sigma_2^4\sigma_3 - 216\sigma_2^3\sigma_3^2 + 144\sigma_2^2\sigma_3^3 + 48\sigma_2\sigma_3^4 - 32\sigma_3^5)] - \frac{1\ 296\gamma_{16}\sigma_2\sigma_3y_2^2y_4}{81\sigma_2^4 - 72\sigma_2^2\sigma_3^2 + 16\sigma_3^4} - \frac{3\gamma_{20}y_2^2y_6}{2\sigma_3^2} + \frac{9\gamma_{16}y_1y_2y_3}{9\sigma_2^2 - 4\sigma_3^2} - \frac{9\gamma_{16}y_5y_6^2}{16\sigma_3} + \frac{\gamma_{20}y_1y_2y_5}{8\sigma_3} - \frac{\gamma_{24}y_1y_2y_6}{24} - \frac{3(3\sigma_2^2 - 2\sigma_2\sigma_3 + 24\sigma_2 + 16\sigma_3)\gamma_{20}y_1y_2y_4}{27\sigma_2^3 - 18\sigma_2^2\sigma_3 - 12\sigma_2\sigma_3^2 + 8\sigma_3^3}, \quad (A5)$$

$$x_6 = -\frac{27\gamma_{16}}{32\sigma_3}y_6^3 + \frac{\gamma_{20}}{2(3\sigma_2 - 2\sigma_3)}y_1^2y_4 - \frac{\gamma_{20}}{96}y_1^2y_5 + \frac{108(3\sigma_2^2 + 4\sigma_3^2)\gamma_{16}y_2^2y_3}{81\sigma_2^4 - 72\sigma_2^2\sigma_3^2 + 16\sigma_3^4} -$$

$$\frac{\gamma_{20}Y_2^2Y_4}{2} [(81\sigma_2^4 - 72\sigma_2^2\sigma_3^2 + 16\sigma_3^4 + 1728\sigma_2^2\sigma_3 - 1152\sigma_2\sigma_3^2 + 10368\sigma_2^2 + 13824\sigma_2\sigma_3 + 4608\sigma_3^2)/(243\sigma_2^5 - 162\sigma_2^4\sigma_3 - 216\sigma_3^3\sigma_2^2 + 144\sigma_2^2\sigma_3^3 + 48\sigma_2\sigma_3^4 - 32\sigma_3^5)] - \frac{\gamma_{20}Y_1Y_2Y_6}{8\sigma_3} + \frac{27\sigma_2\gamma_{16}Y_1Y_2Y_4}{2(9\sigma_2^2 - 4\sigma_3^2)} + \frac{\gamma_{24}Y_1Y_2Y_5}{24} - \frac{2(3\sigma_2\sigma_3 - 2\sigma_3^2 + 36\sigma_2 + 24\sigma_3)\gamma_{20}Y_1Y_2Y_3}{27\sigma_2^3 - 18\sigma_2^2\sigma_3 - 12\sigma_2\sigma_3^2 + 8\sigma_3^3}. \quad (A6)$$

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# Analysis on Nonlinear Dynamics of Circular Truss Antennae in 6D Systems

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**Abstract:** The main tendency of circular truss antennae will be large scale, light weight and high flexibility in future. The circular truss antenna keeps in a folded state during the time of launching. After blastoff, the circular truss antenna unfolds in sequence according to the instruction, saving much space for the satellite. In addition, the caliber of the circular truss antenna can be designed as an ideal value according to requirement. Due to the structural characteristics and the complex spatial environment, the antenna may suffer large-amplitude vibrations, which severely affect the stability of the satellite. The circular truss antenna was simplified as an equivalent cylindrical shell model and the dynamic equations were established. The theoretical analysis and numerical simulation were used to investigate the nonlinear dynamic behaviors of the circular truss antenna in the 6D system. The normal form theory was adopted to simplify the averaged equations. The dynamics of the unperturbed system and the perturbed system was studied. The Shilnikov-type multi-pulse chaotic motion was proved with the energy-phase method, and the effects of the thermal excitation on the nonlinear vibrations of the circular truss antenna system was verified through numerical simulation.

**Key words:** circular truss antenna; normal form; energy phase method; chaotic motion

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