

对称正交铺设矩形叠层板的非线性弯曲*

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摘要

本文应用[1]中提出的奇异摄动方法, 在[3]的基础上, 研究了在各种支承条件下承受均布载荷的对称正交铺设矩形叠层板的非线性弯曲问题, 导出了挠度和应力函数的一致有效的 N 阶形式渐近解. 对承受均布压力, 边界位移为零的简支矩形板进行了分析、计算.

一、问题的提出

随着复合材料在各个部门得到越来越广泛的应用, 对复合材料结构的力学分析也就提出了更迫切的要求. 复合材料的优越性使它更适用于大跨度的结构. 因此, 本文应用江福汝提出的新的构造边界层的方法^[1,2], 在研究了正交异性矩形薄板的非线性弯曲后^[3], 进一步研究了承受均布荷载的对称正交铺设矩形叠层板在各种支承条件下的非线性弯曲, 简化了这类问题^[4,5].

假设所研究的矩形叠层板宽 a , 长 b , 厚度为 t , 承受横向均布载荷 q ; 并设共有 N 层单向增强(正交异性)层. 其材料主方向与层合板坐标轴交错成 0° 和 90° 角. 奇数层的纤维方向是层合板的 x 轴向, 偶数层的纤维方向是层合板的 y 轴向(图1).

(1) 基本方程 对称正交铺设叠层板的本构方程为

$$\begin{Bmatrix} N_x \\ N_y \\ N_{xy} \end{Bmatrix} = \begin{bmatrix} A_{11} & A_{12} & 0 \\ A_{12} & A_{22} & 0 \\ 0 & 0 & A_{66} \end{bmatrix} \begin{Bmatrix} e_x^0 \\ e_y^0 \\ \gamma_{xy}^0 \end{Bmatrix} \quad (1.1)$$

$$\begin{Bmatrix} M_x \\ M_y \\ M_{xy} \end{Bmatrix} = \begin{bmatrix} D_{11} & D_{12} & 0 \\ D_{12} & D_{22} & 0 \\ 0 & 0 & D_{66} \end{bmatrix} \begin{Bmatrix} K_x \\ K_y \\ K_{xy} \end{Bmatrix} \quad (1.2)$$

因此, 中面应变^[6,7]

* 江福汝推荐. 本文曾在1983年“全国奇异摄动理论及应用”学术会议上宣读.

$$\begin{Bmatrix} \varepsilon_x^0 \\ \varepsilon_y^0 \\ \gamma_{xy}^0 \end{Bmatrix} = \begin{bmatrix} A_{11} & A_{12} & 0 \\ A_{12} & A_{22} & 0 \\ 0 & 0 & A_{66} \end{bmatrix}^{-1} \begin{Bmatrix} N_x \\ N_y \\ N_{xy} \end{Bmatrix} = t \begin{bmatrix} A_{11}^* & A_{12}^* & 0 \\ A_{12}^* & A_{22}^* & 0 \\ 0 & 0 & A_{66}^* \end{bmatrix} \begin{Bmatrix} \frac{\partial^2 \varphi}{\partial y^2} \\ \frac{\partial^2 \varphi}{\partial x^2} \\ -\frac{\partial^2 \varphi}{\partial x \partial y} \end{Bmatrix} \quad (1.3)$$

对于常用的叠层板, 刚度 A_{ij} 和 D_{ij} 可用总层数 N , 正交铺设比 M 和主单层刚度比 F 表达⁽⁷⁾. 对称正交铺设叠层板非线性弯曲的 von Kármán 方程为

$$\begin{bmatrix} L_1^* & 0 \\ 0 & L_2^* \end{bmatrix} \begin{Bmatrix} w \\ \varphi \end{Bmatrix} = \begin{Bmatrix} L_3(\varphi, w) + q \\ -L_3(w, w)/2 \end{Bmatrix} \quad (1.4)$$

形中, 算子

$$\left. \begin{aligned} L_1^* &= D_{11} \frac{\partial^4}{\partial x^4} + 2(D_{12} + 2D_{66}) \frac{\partial^4}{\partial x^2 \partial y^2} + D_{22} \frac{\partial^4}{\partial y^4} \\ L_2^* &= A_{22}^* \frac{\partial^4}{\partial x^4} + (2A_{12}^* + A_{66}^*) \frac{\partial^4}{\partial x^2 \partial y^2} + A_{11}^* \frac{\partial^4}{\partial y^4} \end{aligned} \right\} \quad (1.5)$$

$$L_3 = \frac{\partial^2}{\partial x^2} \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial y^2} \frac{\partial^2}{\partial x^2} - 2 \frac{\partial^2}{\partial x \partial y} \frac{\partial^2}{\partial x \partial y} \quad (1.6)$$

引入无量纲量

$$\tilde{w} = \frac{w}{a} \quad \tilde{x} = \frac{x}{a} \quad \tilde{y} = \frac{y}{a} \quad \tilde{\varphi} = \frac{\varphi A_{11}^*}{a^2} \quad \tilde{q} = a A_{11}^* q \quad (1.7)$$

方程(1.4)化为(略去字母上的“~”号)

$$\begin{bmatrix} \varepsilon^2 L_1 & 0 \\ 0 & L_2 \end{bmatrix} \begin{Bmatrix} w \\ \varphi \end{Bmatrix} = \begin{Bmatrix} L_3(\varphi, w) + q \\ -\frac{1}{2} L_3(w, w) \end{Bmatrix} \quad (1.8)$$

式中

$$\varepsilon^2 = \frac{[(F-1)P+1]t^2}{12 \left[\frac{M+F}{1+M} - \frac{1+M}{1+MF} \gamma_{21}^2 \right] a^2} \quad (1.9)$$

$$\left. \begin{aligned} L_1 &= \frac{\partial^4}{\partial x^4} + 2 \frac{D_{12} + 2D_{66}}{D_{11}} \frac{\partial^4}{\partial x^2 \partial y^2} + \frac{D_{22}}{D_{11}} \frac{\partial^4}{\partial y^4} \\ L_2 &= \frac{A_{22}^*}{A_{11}^*} \frac{\partial^4}{\partial x^4} + \frac{2A_{12}^* + A_{66}^*}{A_{11}^*} \frac{\partial^4}{\partial x^2 \partial y^2} + \frac{\partial^4}{\partial y^4} \end{aligned} \right\} \quad (1.10)$$

(2) 边界条件 假设挠度函数 w 的边界条件为

$$w \Big|_{x=0} = f_1(y), \quad \frac{\partial w}{\partial x} \Big|_{x=0} = g_1(y) \quad (1.11)$$

$$w \Big|_{x=1} = f_2(y), \quad \frac{\partial w}{\partial x} \Big|_{x=1} = g_2(y) \quad (1.12)$$

$$w \Big|_{y=0} = f_3(x), \quad -\frac{D_{21}}{D_{22}} \frac{\partial^2 w}{\partial x^2} - \frac{\partial^2 w}{\partial y^2} \Big|_{y=0} = g_3(x) \quad (1.13)$$

$$w \Big|_{y=\frac{b}{a}} = f_4(x), \quad -\frac{D_{21}}{D_{22}} \frac{\partial^2 w}{\partial x^2} - \frac{\partial^2 w}{\partial y^2} \Big|_{y=\frac{b}{a}} = g_4(x) \quad (1.14)$$

应力函数 φ 的边界条件为

$$\left. \frac{\partial^2 \varphi}{\partial y^2} \right|_{x=0} = h_1(y), \quad \left. \frac{\partial^2 \varphi}{\partial x \partial y} \right|_{x=0} = I_1(y) \quad (1.15)$$

$$\left. \frac{\partial^2 \varphi}{\partial y^2} \right|_{x=1} = h_2(y), \quad \left. \frac{\partial^2 \varphi}{\partial x \partial y} \right|_{x=1} = I_2(y) \quad (1.16)$$

$$\left. \frac{\partial^2 \varphi}{\partial x^2} \right|_{y=0} = h_3(x), \quad \left. \frac{\partial^2 \varphi}{\partial x \partial y} \right|_{y=0} = I_3(x) \quad (1.17)$$

$$\left. \frac{\partial^2 \varphi}{\partial x^2} \right|_{y=\frac{b}{a}} = h_4(x), \quad \left. \frac{\partial^2 \varphi}{\partial x \partial y} \right|_{y=\frac{b}{a}} = I_4(x) \quad (1.18)$$

或

$$\int_0^{b/a} t \frac{\partial^2 \varphi}{\partial y^2} dy = \bar{P}_x \frac{b}{a} t \quad (1.19)$$

$$\int_0^1 t \frac{\partial^2 \varphi}{\partial x^2} dx = \bar{P}_y t \quad (1.20)$$

$$\int_0^1 \left[\frac{\partial^2 \varphi}{\partial y^2} + \frac{A_{11}^*}{A_{12}^*} \frac{\partial^2 \varphi}{\partial x^2} - \frac{1}{2} \left(\frac{\partial w}{\partial x} \right)^2 \right] dx = \delta_x \quad (1.21)$$

$$\int_0^{b/a} \left[\frac{A_{21}^*}{A_{11}^*} \frac{\partial^2 \varphi}{\partial y^2} + \frac{A_{22}^*}{A_{11}^*} \frac{\partial^2 \varphi}{\partial x^2} - \frac{1}{2} \left(\frac{\partial w}{\partial y} \right)^2 \right] dy = \delta_y \quad (1.22)$$

式中, $\bar{P}_x \frac{b}{a} t$ 和 $\bar{P}_y t$ 是常数, 为在板平面内, 在 x 方向和在 y 方向的合力; δ_x 和 δ_y 则为板在 x 方向和在 y 方向的伸长.

二、递推方程和边界条件

在边界 $x=0$ 和 $x=1$ 的邻域分别引入边界层函数 $v_n^{(1)}, h_n^{(1)}$ 和 $v_n^{(2)}, h_n^{(2)}$; 在边界 $y=0$ 和 $y=b/a$ 的邻域分别引进边界层函数 $v_n^{(3)}, h_n^{(3)}$ 和 $v_n^{(4)}, h_n^{(4)}$. 假设挠度 w 和应力函数 φ 的 N 阶近似式为

$$\begin{aligned} W_N(x, y, \varepsilon) = & \sum_{n=0}^N \varepsilon^n w_n(x, y) + \sum_{n=0}^N \varepsilon^{n+\alpha_1} v_n^{(1)}(\xi, \eta, y) + \sum_{n=0}^N \varepsilon^{n+\alpha_2} v_n^{(2)}(\xi, \bar{\eta}, y) \\ & + \sum_{n=0}^N \varepsilon^{n+\alpha_3} v_n^{(3)}(x, \alpha, \beta) + \sum_{n=0}^N \varepsilon^{n+\alpha_4} v_n^{(4)}(x, \bar{\alpha}, \bar{\beta}) \end{aligned} \quad (2.1)$$

$$\begin{aligned} \Phi_N(x, y, \varepsilon) = & \sum_{n=0}^N \varepsilon^n \varphi_n(x, y) + \sum_{n=0}^N \varepsilon^{n+\beta_1} h_n^{(1)}(\xi, \eta, y) + \sum_{n=0}^N \varepsilon^{n+\beta_2} h_n^{(2)}(\xi, \bar{\eta}, y) \\ & + \sum_{n=0}^N \varepsilon^{n+\beta_3} h_n^{(3)}(x, \alpha, \beta) + \sum_{n=0}^N \varepsilon^{n+\beta_4} h_n^{(4)}(x, \bar{\alpha}, \bar{\beta}) \end{aligned} \quad (2.2)$$

式中 $\alpha_1, \alpha_2, \alpha_3, \alpha_4$ 和 $\beta_1, \beta_2, \beta_3, \beta_4$ 为待定常数。

将(2.1)式代入边界条件(1.11)~(1.14), 我们看到应取 $\alpha_1 = \alpha_2 = 1, \alpha_3 = \alpha_4 = 2$ 。逐次地比较等式两端 ε 的各次幂的系数, 我们得到关于 $w_n, v_n^{(1)}, v_n^{(2)}, v_n^{(3)}$ 和 $v_n^{(4)}$ 的边界条件^[3]:

$$\left. \begin{aligned} w_0|_{x=0} &= f_1(y) & w_0|_{x=1} &= f_2(y) \\ w_0|_{y=0} &= f_3(x) & w_0|_{y=\frac{b}{a}} &= f_4(x) \\ w_{0,x}|_{x=0} + \delta_{1,0}v_0^{(1)}|_{\eta=0} &= g_1(y) \\ w_{0,x}|_{x=1} + \tilde{\delta}_{1,0}v_0^{(2)}|_{\eta=1} &= g_2(y) \\ -\left\{ \left[w_{0,yy} + \frac{D_{12}}{D_{22}} w_{0,xx} \right]_{y=0} + \gamma_{2,0}v_0^{(3)} \right\}_{\beta=0} &= g_3(x) \\ -\left\{ \left[w_{0,yy} + \frac{D_{12}}{D_{22}} w_{0,xx} \right]_{y=\frac{b}{a}} + \tilde{\gamma}_{2,0}v_0^{(4)} \right\}_{\bar{\beta}=\frac{b}{a}} &= g_4(x) \end{aligned} \right\} \quad (2.3)$$

$$\left. \begin{aligned} w_n|_{x=0} + v_{n-1}^{(1)}|_{\eta=0} &= 0 & w_n|_{x=1} + v_{n-1}^{(2)}|_{\eta=1} &= 0 \\ w_n|_{y=0} + v_{n-2}^{(3)}|_{\beta=0} &= 0 & w_n|_{y=\frac{b}{a}} + v_{n-2}^{(4)}|_{\bar{\beta}=\frac{b}{a}} &= 0 \\ w_{n,x}|_{x=0} + (\delta_{1,0}v_n^{(1)} + \delta_{1,1}v_{n-1}^{(1)})|_{\eta=0} &= 0 \\ w_{n,x}|_{x=1} + (\tilde{\delta}_{1,0}v_n^{(2)} + \tilde{\delta}_{1,1}v_{n-1}^{(2)})|_{\eta=1} &= 0 \\ \left[w_{n,yy} + \frac{D_{12}}{D_{22}} w_{n,xx} \right]_{y=0} + (\gamma_{2,0}v_n^{(3)} + \gamma_{2,1}v_{n-1}^{(3)} + \gamma_{2,2}v_{n-2}^{(3)})_{\beta=0} &= 0 \\ \left[w_{n,yy} + \frac{D_{12}}{D_{22}} w_{n,xx} \right]_{y=\frac{b}{a}} + (\tilde{\gamma}_{2,0}v_n^{(4)} + \tilde{\gamma}_{2,1}v_{n-1}^{(4)} + \tilde{\gamma}_{2,2}v_{n-2}^{(4)})_{\bar{\beta}=\frac{b}{a}} &= 0 \end{aligned} \right\} \quad (2.4)$$

($n=1, 2, \dots, N$)

在上式和以后的计算中, 我们都将带负下标的量取为零。将表达式(2.1)和(2.2)代入(1.8)式, 得到 w_n 和 φ_n 的递推方程

$$L_3(w_0, \varphi_0) = -q, \quad L_2\varphi_0 + \frac{1}{2}L_3(w_0, w_0) = 0 \quad (2.5)$$

$$\left. \begin{aligned} L_3(w_0, \varphi_n) + L_3(w_n, \varphi_0) &= L_1w_{n-2} - \sum_{i=1}^{n-1} L_3(w_i, \varphi_{n-i}) \\ L_2\varphi_n + L_3(w_0, w_n) &= -\frac{1}{2} \sum_{i=1}^{n-1} L_3(w_i, w_{n-i}) \end{aligned} \right\} \quad (2.6)$$

($n=1, 2, \dots, N$)

并且, 我们看到应取 $\beta_1 = \beta_2 = 3, \beta_3 = \beta_4 = 4$ 。于是, 边界层函数 $v_n^{(i)}$ 和 $h_n^{(i)}$ ($i=1, \dots, 4$)的递推方程可以导出(见[3])。最后, 将(2.1)和(2.2)式代入方程(1.15)~(1.22), 得到 $\varphi_n, h_n^{(1)}, h_n^{(2)}, h_n^{(3)}$ 和 $h_n^{(4)}$ 的边界条件

$$\left. \begin{aligned} \varphi_{0,yy}|_{x=0} &= h_1(y) & \varphi_{0,xy}|_{x=0} &= I_1(y) \\ \varphi_{0,yy}|_{x=1} &= h_2(y) & \varphi_{0,xy}|_{x=1} &= I_2(y) \\ \varphi_{0,xx}|_{y=0} &= h_3(x) & \varphi_{0,xy}|_{y=0} &= I_3(x) \\ \varphi_{0,xx}|_{y=\frac{b}{a}} &= h_4(x) & \varphi_{0,xy}|_{y=\frac{b}{a}} &= I_4(x) \end{aligned} \right\} \quad (2.7)$$

$$\left. \begin{aligned}
 &\varphi_{n,yy}|_{x=0} + h_{n-3}^{(1)},_{yy}|_{\eta=0} = 0 \\
 &\varphi_{n,xy}|_{x=0} + [\delta_{1,0}h_{n-2}^{(1)},_y + \delta_{1,1}h_{n-3}^{(1)},_y]|_{\eta=0} = 0 \\
 &\varphi_{n,yy}|_{x=1} + h_{n-3}^{(2)},_{yy}|_{\eta=1} = 0 \\
 &\varphi_{n,xy}|_{x=1} + [\tilde{\delta}_{1,0}h_{n-2}^{(2)},_y + \tilde{\delta}_{1,1}h_{n-3}^{(2)},_y]|_{\eta=1} = 0 \\
 &\varphi_{n,xx}|_{y=0} + h_{n-4}^{(3)},_{xx}|_{\beta=0} = 0 \\
 &\varphi_{n,xy}|_{y=0} + [\gamma_{1,0}h_{n-3}^{(3)},_x + \gamma_{1,1}h_{n-4}^{(3)},_x]|_{\beta=0} = 0 \\
 &\varphi_{n,xx}\Big|_{y=\frac{b}{a} + h_{n-4}^{(4)},_{xx}}\Big|_{\beta=\frac{b}{a}} = 0 \\
 &\varphi_{n,xy}\Big|_{y=\frac{b}{a} + [\tilde{\gamma}_{1,0}h_{n-3}^{(4)},_x + \tilde{\gamma}_{1,1}h_{n-4}^{(4)},_x]}\Big|_{\beta=\frac{b}{a}} = 0
 \end{aligned} \right\} \quad (2.8)$$

(n=1, 2, \dots, N)

或

$$\left. \begin{aligned}
 &\int_0^{b/a} t\varphi_{0,yy}dy = \bar{P}_x \frac{b}{a} t, \quad \int_0^1 t\varphi_{0,xx}dx = \bar{P}_y t \\
 &\int_0^{b/a} \left[\frac{A_{21}^*}{A_{11}^*} \varphi_{0,yy} + \frac{A_{22}^*}{A_{11}^*} \varphi_{0,xx} - \frac{1}{2} (w_{0,y})^2 \right] dy = \delta_y \\
 &\int_0^1 \left[\varphi_{0,yy} + \frac{A_{12}^*}{A_{11}^*} \varphi_{0,xx} - \frac{1}{2} (w_{0,x})^2 \right] dx = \delta_x
 \end{aligned} \right\} \quad (2.9)$$

$$\left. \begin{aligned}
 &\int_0^{b/a} t[\varphi_{n,yy} + h_{n-3}^{(1)},_{yy}]dy = 0, \quad \int_0^{b/a} t[\varphi_{n,yy} + h_{n-3}^{(2)},_{yy}]dy = 0 \\
 &\int_0^1 t[\varphi_{n,xx} + h_{n-4}^{(3)},_{xx}]dx = 0, \quad \int_0^1 t[\varphi_{n,xx} + h_{n-4}^{(4)},_{xx}]dx = 0 \\
 &\int_0^1 \left\{ \varphi_{n,yy} + h_{n-4}^{(3)},_{yy} + \frac{A_{12}^*}{A_{11}^*} (\varphi_{n,xx} + h_{n-4}^{(3)},_{xx}) \right. \\
 &\quad \left. - \frac{1}{2} \left[w_{\frac{n}{2},x}^2 + 2 \sum_{i=0}^n (w_{i,x}w_{n-i,x} + w_{i,x}v_{n-2-i}^{(3)},_x) \right] \right\} dx = 0 \\
 &\int_0^1 \left\{ \varphi_{n,yy} + h_{n-4}^{(4)},_{yy} + \frac{A_{12}^*}{A_{11}^*} (\varphi_{n,xx} + h_{n-4}^{(4)},_{xx}) \right. \\
 &\quad \left. - \frac{1}{2} \left[w_{\frac{n}{2},x}^2 + 2 \sum_{i=0}^n (w_{i,x}w_{n-i,x} + w_{i,x}v_{n-2-i}^{(4)},_x) \right] \right\} dx = 0 \\
 &\int_0^{b/a} \left\{ \frac{A_{21}^*}{A_{11}^*} (\varphi_{n,yy} + h_{n-3}^{(1)},_{yy}) + \frac{A_{22}^*}{A_{11}^*} (\varphi_{n,xx} + h_{n-3}^{(1)},_{xx}) \right. \\
 &\quad \left. - \frac{1}{2} \left[w_{\frac{n}{2},x}^2 + 2 \sum_{i=0}^n (w_{i,y}w_{n-i,y} + w_{i,y}v_{n-1-i}^{(1)},_y) \right] \right\} dy = 0 \\
 &\int_0^{b/a} \left\{ \frac{A_{21}^*}{A_{11}^*} (\varphi_{n,yy} + h_{n-3}^{(2)},_{yy}) + \frac{A_{22}^*}{A_{11}^*} (\varphi_{n,xx} + h_{n-3}^{(2)},_{xx}) \right. \\
 &\quad \left. - \frac{1}{2} \left[w_{\frac{n}{2},y}^2 + 2 \sum_{i=0}^n (w_{i,y}w_{n-i,y} + w_{i,y}v_{n-1-i}^{(2)},_y) \right] \right\} dy = 0
 \end{aligned} \right\} \quad (2.10)$$

(n=1, 2, \dots, N)

三、N阶形式渐近解的导出

方程(2.5)和边界条件(2.3), (2.7), (2.9)给出退化边值问题

$$L_3(w_0, \varphi_0) = -q, \quad L_2\varphi_0 + \frac{1}{2}L_3(w_0, w_0) = 0 \quad (3.1)$$

$$\left. \begin{aligned} w_0|_{x=0} &= f_1(y), & w_0|_{x=1} &= f_2(y) \\ w_0|_{y=0} &= f_3(x), & w_0|_{y=\frac{b}{a}} &= f_4(x) \end{aligned} \right\} \quad (3.2)$$

$$\left. \begin{aligned} \varphi_{0,yy}|_{x=0} &= h_1(y), & \varphi_{0,yy}|_{x=1} &= h_2(y) \\ \varphi_{0,xx}|_{y=0} &= h_3(x), & \varphi_{0,xx}|_{y=\frac{b}{a}} &= h_4(x) \end{aligned} \right\} \quad (3.3)$$

或

$$\left. \begin{aligned} \int_0^{b/a} t\varphi_{0,yy}dy &= \bar{P}_x \frac{b}{a}t, & \int_0^1 t\varphi_{0,xx}dx &= \bar{P}_y t \\ \int_0^1 \left[\varphi_{0,yy} + \frac{A_{12}^*}{A_{11}^*}\varphi_{0,xx} - \frac{1}{2}w_{0,x}^2 \right] dx &= \delta_x \\ \int_0^{b/a} \left[\frac{A_{21}^*}{A_{11}^*}\varphi_{0,yy} + \frac{A_{22}^*}{A_{11}^*}\varphi_{0,xx} - \frac{1}{2}w_{0,y}^2 \right] dy &= \delta_y \end{aligned} \right\} \quad (3.4)$$

解此边值问题, 将求得的薄膜解 w_0, φ_0 代入 $v_0^{(i)}$ 及 $h_0^{(i)}$ ($i=1, \dots, 4$) 之方程^[3], 利用边界条件(2.3), 在 $h_1(y) > 0$, $h_2(y) > 0$ 和 $h_3(x) > 0$, $h_4(x) > 0$ 的条件下, 得到边界层函数

$$\left. \begin{aligned} v_0^{(1)}(\xi, \eta, y) &= C_0^{(1)}(\eta, y)e^{-\xi} = C_0^{(1)}(\eta, y)\exp\left[-\frac{1}{\varepsilon}\int_0^x \sqrt{\varphi_{0,yy}(x, y)} dx\right] \\ v_0^{(2)}(\xi, \bar{\eta}, y) &= C_0^{(2)}(\bar{\eta}, y)e^{-\xi} = C_0^{(2)}(\bar{\eta}, y)\exp\left[-\frac{1}{\varepsilon}\int_x^1 \sqrt{\varphi_{0,yy}(x, y)} dx\right] \\ v_0^{(3)}(x, \alpha, \beta) &= C_0^{(3)}(x, \beta)\exp\left[-\frac{\alpha}{\sqrt{e_1}}\right] = C_0^{(3)}(x, \beta)\exp\left[-\frac{1}{\varepsilon\sqrt{e_1}}\int_0^y \sqrt{\varphi_{0,xx}(x, y)} dy\right] \\ v_0^{(4)}(x, \bar{\alpha}, \bar{\beta}) &= C_0^{(4)}(x, \bar{\beta})\exp\left[-\frac{\bar{\alpha}}{\sqrt{e_1}}\right] = C_0^{(4)}(x, \bar{\beta})\exp\left[-\frac{1}{\varepsilon\sqrt{e_1}}\int_y^{b/a} \sqrt{\varphi_{0,xx}(x, y)} dy\right] \end{aligned} \right\} \quad (3.5)$$

$$\left. \begin{aligned} h_0^{(1)}(\xi, \eta, y) &= -A^{-1}(\bar{\eta}, y)w_{0,yy}C_0^{(1)}(\eta, y)e^{-\xi} \\ h_0^{(2)}(\xi, \bar{\eta}, y) &= -A^{-1}(\bar{\eta}, y)w_{0,yy}C_0^{(2)}(\bar{\eta}, y)e^{-\xi} \\ h_0^{(3)}(x, \alpha, \beta) &= -e_1^{-1}B^{-1}(x, \alpha)w_{0,xx}C_0^{(3)}(x, \alpha)e^{-\alpha} \\ h_0^{(4)}(x, \bar{\alpha}, \bar{\beta}) &= -e_1^{-1}B^{-1}(x, \bar{\alpha})w_{0,xx}C_0^{(4)}(x, \bar{\alpha})e^{-\bar{\alpha}} \end{aligned} \right\} \quad (3.6)$$

式中 $e_1 = D_{22}/D_{11}$, $A(x, y) = \varphi_{0,yy}$, $B(x, y) = \varphi_{0,xx}$; $C_0^{(1)}(\eta, y)$, \dots , $C_0^{(4)}(x, \bar{\beta})$ 为待定函数, 其边值条件为

$$\left. \begin{aligned} C_0^{(1)}(\eta, y) \Big|_{\eta=0} &= -\frac{g_1(y) - w_{0,x}(0, y)}{\sqrt{h_1(y)}}, \quad C_0^{(2)}(\tilde{\eta}, y) \Big|_{\tilde{\eta}=1} = -\frac{g_2(y) - w_{0,x}(1, y)}{\sqrt{h_2(y)}} \\ C_0^{(3)}(x, \beta) \Big|_{\beta=0} &= -e_1 h_3^{-1}(x) \left\{ g_3(x) + \left[w_{0,yy}(x, 0) + \frac{D_{12}}{D_{22}} w_{0,xx}(x, 0) \right] \right\} \\ C_0^{(4)}(x, \tilde{\beta}) \Big|_{\tilde{\beta}=\frac{b}{a}} &= -e_1 h_4^{-1}(x) \left\{ g_4(x) + \left[w_{0,yy}\left(x, \frac{b}{a}\right) + \frac{D_{12}}{D_{22}} w_{0,xx}\left(x, \frac{b}{a}\right) \right] \right\} \end{aligned} \right\} \quad (3.7)$$

在(2.6), (2.4), (2.8)和(2.10)中, 令 $n=1$, 得到 w_1 和 φ_1 的线性边值问题

$$L_3(w_0, \varphi_1) + L_3(w_1, \varphi_0) = 0, \quad L_2\varphi_1 + L_3(w_0, w_1) = 0 \quad (3.8)$$

$$\left. \begin{aligned} w_1 \Big|_{x=0} &= -v_0^{(1)} \Big|_{\eta=0} = -C_0^{(1)}(0, y) \\ w_1 \Big|_{x=1} &= -v_0^{(2)} \Big|_{\tilde{\eta}=1} = -C_0^{(2)}(1, y) \\ w_1 \Big|_{y=0} &= 0, \quad w_1 \Big|_{y=\frac{b}{a}} = 0 \end{aligned} \right\} \quad (3.9)$$

$$\left. \begin{aligned} \varphi_{1,yy} \Big|_{x=0} &= 0, \quad \varphi_{1,yy} \Big|_{x=1} = 0 \\ \varphi_{1,xx} \Big|_{y=0} &= 0, \quad \varphi_{1,xx} \Big|_{y=\frac{b}{a}} = 0 \end{aligned} \right\} \quad (3.10)$$

或

$$\left. \begin{aligned} \int_0^{b/a} t \varphi_{1,yy} dy &= 0, \quad \int_0^1 t \varphi_{1,xx} dx = 0 \\ \int_0^1 \left[\varphi_{1,yy} + \frac{A_{12}^*}{A_{11}^*} \varphi_{1,xx} - w_{0,x} w_{1,x} - w_{0,x} v_0^{(1)} \right] dx &= 0 \\ \int_0^{b/a} \left[\frac{A_{21}^*}{A_{11}^*} \varphi_{1,yy} + \frac{A_{22}^*}{A_{11}^*} \varphi_{1,xx} - w_{0,y} w_{1,x} - w_{0,y} v_0^{(1)} \right] dy &= 0 \end{aligned} \right\} \quad (3.11)$$

将 $w_0, \varphi_0, v_0^{(1)}, \dots, v_0^{(4)}$ 和 $h_0^{(1)}, \dots, h_0^{(4)}$ 代入以上方程, 求得 w_1 和 φ_1 后, 代入 $v_n^{(4)}$ 的公式^[3], 并令各等式右端为零, 得到确定 $C_0^{(1)}(\eta, y), \dots, C_0^{(4)}(x, \tilde{\beta})$ 的一阶线性偏微分方程

$$\left. \begin{aligned} 2A \frac{\partial C_0^{(1)}}{\partial \eta} + 2\varphi_{0,xy} \frac{\partial C_0^{(1)}}{\partial y} + \left[\frac{5}{2} A_{,x} + \varphi_{1,yy} A^{\frac{1}{2}} \right] C_0^{(1)} &= 0 \\ 2A \frac{\partial C_0^{(2)}}{\partial \tilde{\eta}} + 2\varphi_{0,xy} \frac{\partial C_0^{(2)}}{\partial y} + \left[\frac{5}{2} A_{,x} + \varphi_{1,yy} A^{\frac{1}{2}} \right] C_0^{(2)} &= 0 \\ 2B \frac{\partial C_0^{(3)}}{\partial \beta} + 2\varphi_{0,xy} \frac{\partial C_0^{(3)}}{\partial x} + \left[\frac{5}{2} B_{,y} + \varphi_{1,xx} \left(\frac{B}{e_1} \right)^{\frac{1}{2}} \right] C_0^{(3)} &= 0 \\ 2B \frac{\partial C_0^{(4)}}{\partial \tilde{\beta}} + 2\varphi_{0,xy} \frac{\partial C_0^{(4)}}{\partial x} + \left[\frac{5}{2} B_{,y} + \varphi_{1,xx} \left(\frac{B}{e_1} \right)^{\frac{1}{2}} \right] C_0^{(4)} &= 0 \end{aligned} \right\} \quad (3.12)$$

在 $A(x, y) > 0$ 和 $B(x, y) > 0$ 的条件下, 由Cauchy条件(3.7)可以唯一地求得 $C_0^{(1)}(\eta, y), \dots, C_0^{(4)}(x, \tilde{\beta})$. 至此, 边界层函数 $v_0^{(1)}, \dots, v_0^{(4)}$; $h_0^{(1)}, \dots, h_0^{(4)}$ 完全确定. 同样运算下去, 我们可逐步地求得展开式(2.1)和(2.2)中的 $w_n, v_n^{(1)}, \dots, v_n^{(4)}$ 和 $\varphi_n, h_n^{(1)}, \dots, h_n^{(4)}$ ($n=1, 2, \dots, N$).

四、承受均布法向压力，边界位移为零的简支矩形板

我们以承受均布法向压力 q 和边缘载荷 $\bar{P}_x \frac{b}{a} t$ 、 $\bar{P}_y t$ ，使边界位移为零的简支、正交铺设矩形叠层板（图1）为例进行具体的分析和讨论。

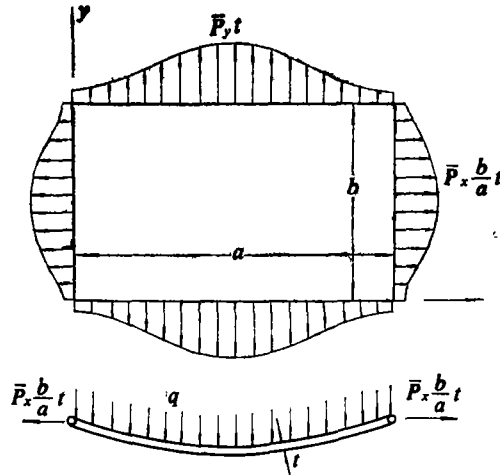


图 1

板的（无量纲）方程为

$$\left. \begin{aligned} e^2 L_1 w - L_3(w, \varphi) &= q \\ L_2 \varphi + L_3(w, w) / 2 &= 0 \end{aligned} \right\} \quad (4.1)$$

边界条件为

$$\left. \begin{aligned} w \Big|_{x=0} = w \Big|_{x=1} = w \Big|_{y=0} = w \Big|_{y=b/a} = 0 \\ \frac{\partial^2 w}{\partial x^2} \Big|_{x=0} = \frac{\partial^2 w}{\partial x^2} \Big|_{x=1} = \frac{\partial^2 w}{\partial y^2} \Big|_{y=0} = \frac{\partial^2 w}{\partial y^2} \Big|_{y=b/a} = 0 \end{aligned} \right\} \quad (4.2)$$

$$\left. \begin{aligned} \int_0^{b/a} t \frac{\partial^2 \varphi}{\partial y^2} dy &= \bar{P}_x \frac{b}{a} t, \quad \int_0^1 t \frac{\partial^2 \varphi}{\partial x^2} dx = \bar{P}_y t \\ \int_0^1 \left[\frac{\partial^2 \varphi}{\partial y^2} + \frac{A_{12}^*}{A_{11}^*} \frac{\partial^2 \varphi}{\partial x^2} - \frac{1}{2} \left(\frac{\partial w}{\partial x} \right)^2 \right] dx &= 0 \\ \int_0^{b/a} \left[\frac{A_{21}^*}{A_{11}^*} \frac{\partial^2 \varphi}{\partial y^2} + \frac{A_{22}^*}{A_{11}^*} \frac{\partial^2 \varphi}{\partial x^2} - \frac{1}{2} \left(\frac{\partial w}{\partial y} \right)^2 \right] dy &= 0 \end{aligned} \right\} \quad (4.3)$$

展开式(2.1)和(2.2)中的首项 w_0, φ_0 确定于以下退化边值问题($e=0$)

$$L_3(w_0, \varphi_0) = -q, \quad L_2 \varphi_0 = L_3(w_0, w_0) / 2 \quad (4.4)$$

$$w_0 \Big|_{x=0} = w_0 \Big|_{x=1} = w_0 \Big|_{y=0} = w_0 \Big|_{y=b/a} = 0 \quad (4.5)$$

$$\int_0^{b/a} t \varphi_{0,yy} dy = \bar{P}_x \frac{b}{a} t, \quad \int_0^1 t \varphi_{0,xx} dx = \bar{P}_y t \quad (4.6)$$

$$\left. \begin{aligned} \int_0^1 \left[\varphi_{0,yy} + \frac{A_{12}^*}{A_{11}^*} \varphi_{0,xx} - \frac{1}{2} (w_{0,z})^2 \right] dx = 0 \\ \int_0^{b/a} \left[\frac{A_{21}^*}{A_{11}^*} \varphi_{0,yy} + \frac{A_{22}^*}{A_{11}^*} \varphi_{0,xx} - \frac{1}{2} (w_{0,y})^2 \right] dy = 0 \end{aligned} \right\} \quad (4.7)$$

已知均布法向压力可以表为

$$q = \sum_{r=1,3,5}^{\infty} \sum_{s=1,3,5}^{\infty} q_{rs} \sin r\pi x \sin \frac{as\pi}{b} y \quad (4.8)$$

式中 $q_{rs} = 1/rs \cdot (4/\pi)^2 q_0$, 我们假设挠度函数为

$$w_0 = \sum_{m=1,3,5}^{\infty} \sum_{n=1,3,5}^{\infty} w_0^{mn} \sin m\pi x \sin \frac{an\pi}{b} y \quad (4.9)$$

则边界条件(4.5)自然满足. 为满足方程(4.4)和边界条件(4.6), 选取应力函数

$$\varphi_0 = \frac{\bar{P}_x y^2}{2} + \frac{\bar{P}_y x^2}{2} + \sum_{m=0,2,4}^{\infty} \sum_{n=0,2,4}^{\infty} \varphi_0^{mn} \cos m\pi x \cos \frac{an\pi}{b} y \quad (4.10)$$

边界位移为零的条件(4.7)化为

$$\left. \begin{aligned} \bar{P}_x + \frac{A_{12}^*}{A_{11}^*} \bar{P}_y = \frac{\pi^2}{8} \sum_{m=1,3,5}^{\infty} \sum_{n=1,3,5}^{\infty} m^2 (w_0^{mn})^2 \\ \frac{A_{21}^*}{A_{11}^*} \bar{P}_x + \frac{A_{22}^*}{A_{11}^*} \bar{P}_y = \frac{a^2 \pi^2}{8b^2} \sum_{m=1,3,5}^{\infty} \sum_{n=1,3,5}^{\infty} n^2 (w_0^{mn})^2 \end{aligned} \right\} \quad (4.11)$$

将(4.9)和(4.10)式代入方程(4.4)的第二式, 比较方程两边同类三角项的系数, 得到 φ_0^{mn} 和 w_0^{mn} 之间的关系为

$$\varphi_0^{mn} = \frac{1}{4 \left(c_2 m^4 + d_2 \frac{m^2 n^2 a^2}{b^2} + e_2 \frac{n^4 a^4}{b^4} \right)} \sum a_{r,s,q} w_0^{rs} w_0^{sq} \quad (4.12)$$

式中求和号包括了 $w_0^{rs} w_0^{sq}$ 对于 $m=r \pm p, n=s \pm q$ 的所有乘积, 系数

$$a_{r,s,q} = 2rs pq \pm (r^2 q^2 + s^2 p^2) \quad (4.13)$$

当 $m=0$ 或 $n=0$ 且 $r=q, s=p$ 时, 应将上式除以 2; 当 $m=r+p, n=s-q$ 或 $m=r-p, n=s+q$ 时, 上式括号前的符号取正号, 否则取负号.

再将(4.8), (4.9)和(4.10)式代入方程(4.4)的第一式, 比较方程两边同类三角项的系数, 得到挠度、应力函数与法向载荷的系数 w_0^{mn}, φ_0^{mn} 和 q_{mn} 之间的关系式, 即

$$q_{mn} = \bar{P}_x \cdot m^2 \pi^2 w_0^{mn} + \bar{P}_y \frac{a^2 n^2 \pi^2}{b^2} w_0^{mn} + \frac{a^2 \pi^4}{4b^2} \sum b_{r,s,q} \varphi_0^{rs} w_0^{sq} \quad (4.14)$$

式中求和号包括了 $\varphi_0^{rs} w_0^{sq}$ 对于 $m=r \pm p, n=s \pm q$ 的所有乘积, 系数

$$b_{r,s,q} = \pm (rq \pm sp)^2 \quad (\text{当 } r \neq 0, s \neq 0) \quad (4.15)$$

当 $r=0$ 或 $s=0$ 时应将上式乘以 2; 当 $r-p=m$ 或 $s-q=n$ (但非同时), 则括号外的符号取正号, 否则取负号. 当 $m=r+p, n=s-q$ 或 $m=r-p, n=s+q$ 时, 括号内的符号取正号, 否则为负.

将关系式(4.12)代入(4.14)式, 便得到关于挠度系数 w_0^{00} 的三次方程组, 解此方程组. 求得 w_0^{00} 后再代回(4.12)式, 就可计算应力函数系数 φ_0^{00} .

在(4.8)和(4.9)式中若取两项计算, 即假设

$$w_0 = w_0^{01} \sin \pi x \sin \frac{a\pi}{b} y + w_0^{31} \sin 3\pi x \sin \frac{a\pi}{b} y \quad (4.16)$$

$$q = \frac{16}{\pi^2} q_0 \sin \pi x \sin \frac{a\pi}{b} y + \frac{16}{3\pi^2} q_0 \sin 3\pi x \sin \frac{a\pi}{b} y \quad (4.17)$$

选取

$$\begin{aligned} \varphi_0 = & \frac{\bar{P}_x y^2}{2} + \frac{\bar{P}_y x^2}{2} + \varphi_0^{20} \cos 2\pi x + \varphi_0^{02} \cos \frac{2a\pi}{b} y + \varphi_0^{22} \cos 2\pi x \cos \frac{2a\pi}{b} y \\ & + \varphi_0^{40} \cos 4\pi x + \varphi_0^{42} \cos 4\pi x \cos \frac{2a\pi}{b} y + \varphi_0^{60} \cos 6\pi x \end{aligned} \quad (4.18)$$

则由(4.12)式, 有

$$\left. \begin{aligned} \varphi_0^{20} &= \frac{a^2}{32c_2 b^2} [(w_0^{11})^2 - 2w_0^{11} w_0^{31}], \quad \varphi_0^{02} = \frac{b^2}{32d_2 a^2} [(w_0^{11})^2 + 9(w_0^{31})^2] \\ \varphi_0^{22} &= \frac{w_0^{11} w_0^{31}}{4 \left[c_2 \frac{b^2}{a^2} + d_2 + e_2 \frac{a^2}{b^2} \right]}, \quad \varphi_0^{42} = - \frac{w_0^{11} w_0^{31}}{16 \left[16c_2 \frac{b^2}{a^2} + 4d_2 + e_2 \frac{a^2}{b^2} \right]} \\ \varphi_0^{40} &= \frac{a^2}{64c_2 b^2} w_0^{31} w_0^{11}, \quad \varphi_0^{60} = \frac{a^2}{288c_2 b^2} (w_0^{31})^2 \end{aligned} \right\} \quad (4.19)$$

然后, 由(4.14)式, 我们得到

$$\left. \begin{aligned} \frac{16q_0}{\pi^2} &= \pi^2 \bar{P}_x w_0^{11} + \frac{a^2 \pi^2}{b^2} \bar{P}_y w_0^{11} + \frac{\pi^4}{16} \left(\frac{a^4}{c_2 b^4} + \frac{1}{d_2} \right) (w_0^{11})^3 + \frac{3a^4 \pi^4}{16c_2 b^4} (w_0^{11})^2 w_0^{31} \\ &\quad - \frac{\pi^4}{16} \left[\frac{4a^4}{c_2 b^4} + \frac{9}{d_2} + \frac{16}{\left(c_2 \frac{b^4}{a^4} + d_2 \frac{b^2}{a^2} + e_2 \right)} + \frac{1}{\left(16c_2 \frac{b^4}{a^4} + 4d_2 \frac{b^2}{a^2} + e_2 \right)} \right] w_0^{11} \cdot (w_0^{31})^2 \\ \frac{16q_0}{3\pi^2} &= 9\pi^2 \bar{P}_x w_0^{31} + \frac{a^2 \pi^2}{b^2} \bar{P}_y w_0^{31} + \frac{a^4 \pi^4}{16c_2 b^4} (w_0^{11})^3 - \frac{\pi^4}{16} \left(\frac{81}{d_2} + \frac{a^4}{c_2 b^4} \right) (w_0^{31})^3 \\ &\quad - \frac{\pi^4}{16} \left[\frac{4a^4}{c_2 b^4} + \frac{9}{d_2} + \frac{16}{\left(c_2 \frac{b^4}{a^4} + d_2 \frac{b^2}{a^2} + e_2 \right)} + \frac{1}{\left(16c_2 \frac{b^4}{a^4} + 4d_2 \frac{b^2}{a^2} + e_2 \right)} \right] (w_0^{11})^2 \cdot w_0^{31} \end{aligned} \right\} \quad (4.20)$$

边界条件(4.11)为

$$\left. \begin{aligned} \bar{P}_x + \frac{A_{11}^*}{A_{11}^*} \bar{P}_y &= \frac{\pi^2}{8} [(w_0^{11})^2 + 9(w_0^{31})^2] \\ \frac{A_{21}^*}{A_{11}^*} \bar{P}_x + \frac{A_{22}^*}{A_{11}^*} \bar{P}_y &= \frac{a^2 \pi^2}{8b^2} [(w_0^{11})^2 + (w_0^{31})^2] \end{aligned} \right\} \quad (4.21)$$

联解以上方程, 求得 \bar{P}_x 、 \bar{P}_y 与 w_0^{11} 、 w_0^{31} 的关系后, 代入方程(4.20), 联解该方程组, 即得 w_0^{11} 和 w_0^{31} ; 再代入(4.19)式, 便计算得 φ_0^{20} , φ_0^{02} , \dots , φ_0^{60} .

由(3.6)和(3.7)式, 可知边界层型函数

$$v_0^{(1)} = v_0^{(2)} = v_0^{(3)} = v_0^{(4)} = 0, \quad h_0^{(1)} = h_0^{(2)} = h_0^{(3)} = h_0^{(4)} = 0 \quad (4.22)$$

w_1 和 φ_1 确定于边值问题

$$L_2(w_1, \varphi_0) + L_2(w_0, \varphi_1) = 0, \quad L_2\varphi_1 + L_2(w_0, w_1) = 0 \quad (4.23)$$

$$\left. \begin{aligned} \int_0^{b/a} t\varphi_{1,yy}dy &= 0 & w_1|_{x=0} &= w_1|_{x=1} = 0 \\ \int_0^1 t\varphi_{1,xx}dx &= 0 & w_1|_{y=0} &= w_1|_{y=b/a} = 0 \\ \int_0^1 \left[\varphi_{1,yy} + \frac{A_{11}^*}{A_{11}^*} \varphi_{1,xx} - w_{0,x}w_{1,x} - w_{0,x}v_{0,x}^{(1)} \right] dx &= 0 \\ \int_0^{b/a} \left[\frac{A_{21}^*}{A_{11}^*} \varphi_{1,yy} + \frac{A_{22}^*}{A_{11}^*} \varphi_{1,xx} - w_{0,y}w_{1,y} - w_{0,y}v_{0,y}^{(1)} \right] dy &= 0 \end{aligned} \right\} \quad (4.24)$$

显然, $w_1 \equiv 0, \varphi_1 \equiv 0, v_i^{(i)} = h_i^{(i)} = 0 \quad (i=1, \dots, 4)$.

w_2 和 φ_2 确定于边值问题

$$\left. \begin{aligned} L_3(w_0, \varphi_2) + L_3(w_2, \varphi_0) &= L_1w_0 \\ L_2\varphi_2 + L_3(w_0, w_2) &= 0 \end{aligned} \right\} \quad (4.25)$$

$$\left. \begin{aligned} w_2|_{x=0} &= w_2|_{x=1} = 0, \quad w_2|_{y=0} = w_2|_{y=b/a} = 0, \quad \int_0^{b/a} t\varphi_{2,yy}dy = 0, \quad \int_0^1 t\varphi_{2,xx}dx = 0 \\ \int_0^1 \left\{ \left[\varphi_{2,yy} + \frac{A_{12}^*}{A_{11}^*} \varphi_{2,xx} \right] - \frac{1}{2} \left[w_{1,x}^2 + 2(w_{1,x}v_{0,x}^{(1)} + w_{0,x}v_{1,x}^{(1)} + w_{0,x}v_{0,x}^{(2)}) \right] \right\} dx &= 0 \\ \int_0^{b/a} \left\{ \left[\frac{A_{21}^*}{A_{11}^*} \varphi_{2,yy} + \frac{A_{22}^*}{A_{11}^*} \varphi_{2,xx} \right] - \frac{1}{2} \left[w_{1,y}^2 + 2(w_{1,y}v_{0,y}^{(1)} + w_{0,y}v_{1,y}^{(1)} + w_{0,y}v_{0,y}^{(2)}) \right] \right\} dy &= 0 \end{aligned} \right\} \quad (4.26)$$

假设挠度和应力函数分别为

$$w_2 = w_2^{11} \sin \pi x \sin \frac{a\pi}{b} y + w_2^{31} \sin 3\pi x \sin \frac{a\pi}{b} y \quad (4.27)$$

$$\begin{aligned} \varphi_2 = \varphi_2^{20} \cos 2\pi x + \varphi_2^{01} \cos \frac{a\pi}{b} y + \varphi_2^{11} \cos 2\pi x \cos \frac{2a\pi}{b} y + \varphi_2^{10} \cos 4\pi x \\ + \varphi_2^{12} \cos 4\pi x \cos \frac{2a\pi}{b} y + \varphi_2^{10} \cos 6\pi x \end{aligned} \quad (4.28)$$

由(4.12)式求得

$$\left. \begin{aligned} \varphi_2^{20} &= \frac{a^2}{16c_2b^2} (w_0^{11} w_2^{11} - w_0^{31} w_2^{31} - w_0^{31} w_2^{11}) \\ \varphi_2^{02} &= \frac{b^2}{16d_2a^2} (w_0^{11} w_2^{11} + 9w_0^{31} w_2^{31}) \\ \varphi_2^{22} &= \frac{w_0^{11}w_2^{31} + w_0^{31}w_2^{11}}{4(c_2 \frac{b^2}{a^2} + d_2 + e_2 \frac{a^2}{b^2})}, \quad \varphi_2^{40} = \frac{a^2}{64c_2b^2} (w_0^{11} w_2^{31} + w_0^{31} w_2^{11}) \\ \varphi_2^{42} &= - \frac{w_0^{11}w_2^{31} + w_0^{31}w_2^{11}}{16(16c_2 \frac{b^2}{a^2} + 4d_2 + e_2 \frac{a^2}{b^2})}, \quad \varphi_2^{60} = \frac{a^2}{144c_2b^2} w_0^{31} w_2^{31} \end{aligned} \right\} \quad (4.29)$$

再代入方程(4.14)

$$\begin{aligned}
 & -2(w_0^{11} \varphi_2^{02} + w_2^{11} \varphi_0^{02}) - 2(w_0^{11} \varphi_0^{20} + w_2^{11} \varphi_0^{20}) - 4(w_0^{31} \varphi_2^{22} + w_2^{31} \varphi_0^{22}) \\
 & + 2(w_0^{31} \varphi_2^{20} + w_2^{31} \varphi_0^{20}) + (w_0^{31} \varphi_2^{42} + w_2^{31} \varphi_0^{42}) - 8(w_0^{31} \varphi_2^{40} + w_2^{31} \varphi_0^{40}) \\
 & = w_0^{11} \left(c_1 \frac{b^2}{a^2} + d_1 + e_1 \frac{a^2}{b^2} \right) \\
 & 2(w_0^{11} \varphi_2^{20} + w_2^{11} \varphi_0^{20}) - 18(w_0^{31} \varphi_0^{02} + w_2^{31} \varphi_2^{02}) - 4(w_0^{11} \varphi_2^{22} + w_2^{11} \varphi_0^{22}) \\
 & - 8(w_0^{11} \varphi_2^{40} + w_2^{11} \varphi_0^{40}) + (w_0^{11} \varphi_2^{42} + w_2^{11} \varphi_0^{42}) - 18(w_0^{31} \varphi_2^{20} + w_2^{31} \varphi_0^{20}) \\
 & = w_0^{31} \left(81c_1 \frac{b^2}{a^2} + 9d_1 + e_1 \frac{a^2}{b^2} \right)
 \end{aligned} \tag{4.30}$$

即可解得 w_2^{11} , w_2^{31} , 进一步可求得 $\varphi_2^{20}, \dots, \varphi_2^{80}$.

于是, 挠度和应力函数的二阶近似式分别为

$$W = (w_0^{11} + \varepsilon^2 w_2^{11}) \sin \pi x \sin \frac{a\pi}{b} y + (w_0^{31} + \varepsilon^2 w_2^{31}) \sin 3\pi x \sin \frac{a\pi}{b} y + O(\varepsilon^2) \tag{4.31}$$

$$\begin{aligned}
 \Phi = & \frac{\bar{P}_x y^2}{2} + \frac{\bar{P}_y x^2}{2} + (\varphi_0^{20} + \varepsilon^2 \varphi_2^{20}) \cos 2\pi x + (\varphi_0^{02} + \varepsilon^2 \varphi_2^{02}) \cos \frac{2a\pi}{b} y \\
 & + (\varphi_0^{22} + \varepsilon^2 \varphi_2^{22}) \cos 2\pi x \cos \frac{2a\pi}{b} y + (\varphi_0^{40} + \varepsilon^2 \varphi_2^{40}) \cos 4\pi x \\
 & + (\varphi_0^{42} + \varepsilon^2 \varphi_2^{42}) \cos 4\pi x \cos \frac{2a\pi}{b} y + (\varphi_0^{80} + \varepsilon^2 \varphi_2^{80}) \cos 6\pi x + O(\varepsilon^2)
 \end{aligned} \tag{4.32}$$

下面, 我们以高模量石墨/环氧复合材料为例作具体数值计算.

高模量石墨/环氧性能为

$$\begin{aligned}
 E_1 &= 137.9 \times 10^8 \text{kpa} (20.0 \times 10^6 \text{psi}) \\
 E_2 &= E_3 = 14.48 \times 10^8 \text{kpa} (2.1 \times 10^6 \text{psi}) \\
 G_{12} &= G_{23} = G_{31} = 4.98 \times 10^8 \text{kpa} (0.85 \times 10^6 \text{psi}) \\
 \nu_{12} &= \nu_{23} = \nu_{31} = 0.21
 \end{aligned}$$

对于 $N=1$ (单层), 有

$$\varepsilon = 0.289346,$$

$$c_1 = 1, \quad e_1 = \frac{D_{22}}{D_{11}} = 0.105, \quad d_1 = \frac{2D_3}{D_{11}} = 0.213313$$

$$c_2 = \frac{E_1}{E_2} = 9.523809, \quad e_2 = 1, \quad d_2 = \frac{E_1}{G_{12}} - 2\nu_{12} = 23.109412$$

对于 $N=3$, 有

$$M = 2, \quad F = \frac{E_2}{E_1} = 0.105, \quad P = \frac{1}{27}, \quad \varepsilon = 0.339155$$

$$c_1 = 1, \quad e_1 = \frac{D_{22}}{D_{11}} = 0.142885, \quad d_1 = \frac{2(D_{12} + 2D_{00})}{D_{11}} = 0.220626$$

$$c_2 = \frac{A_{12}^*}{A_{11}^*} = 1.739669, \quad e_2 = 1, \quad d_2 = \frac{2A_{12}^* + A_{66}^*}{A_{11}^*} = 16.448774$$

我们将 $N=1$ (单层板) 和 $N=3$ 的叠层板的 (无量纲) 载荷与中心挠度的关系曲线图对长宽比取几种不同比值的情况分别作于图 2 中。与各向同性板相同^[4], 在给定的压力下, 挠度随长宽比值的减少而增加; 同时, 我们看到叠层板的刚度较之单层板大为提高。在叠层板中心和角点处的最外层纤维的弯曲应力则示于图 3 中。

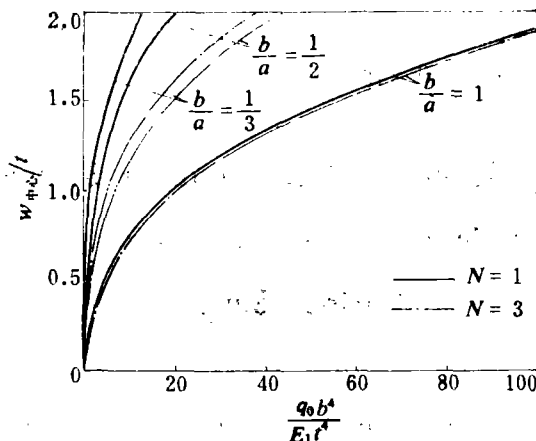


图 2

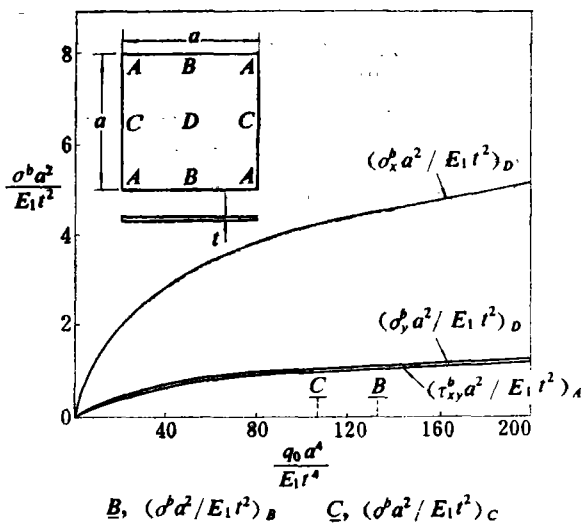


图 3

因为参数 e 是小的, 所以挠度 w 和应力函数 φ 的 N 阶渐近表达式很快收敛。虽然我们在 (4.9) 式中只取两项计算, 且只作到二阶近似, 结果是令人满意的。其中心挠度的近似结果仅比板的 von Kármán 大挠度方程的解低 0.126%。

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Nonlinear Bendings of Rectangular Symmetrically Laminated Cross-Ply Plates under Various Supports

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Abstract

This paper studies the nonlinear bendings of rectangular symmetrically laminated cross-ply plates subjected to uniform pressure under various supports on the basis of [3] by the singular perturbation method offered in [1]. The uniformly valid N -order asymptotic solutions of the deflection and stress function are derived. Analyses and numerical solutions are given for simply supported rectangular laminates, whose edge displacement is vanished.